

# Who we are

GGB helps create a world of motion with minimal frictional loss through plain bearing and surface engineering technologies. With R&D, testing and production facilities in the United States, Germany, France, Brazil, Slovakia and China, GGB partners with customers worldwide on customized tribological design solutions that are efficient and environmentally sustainable. GGB's engineers bring their expertise and passion for tribology to a wide range of industries, including automotive, aerospace and industrial manufacturing. To learn more about tribology for surface engineering from GGB, visit www.ggbearings.com.

Our products are used in tens of thousands of critical applications every day on our planet. It is always our goal to provide superior, high-quality solutions for our customers' needs, no matter where those demands take our products. From space vehicles to golf carts and virtually everything in between; we offer the industry's most extensive range of high performance, maintenance-free bearing solutions for a multitude of applications:



**Agriculture** 

**Energy** 





**Aerospace** 



**E-Mobility** 



**Exoskeletons** 

**Automotive** 



**Fluid Power** 

**Industrial** 



**Mining** 



**Primary Metals** 





**Railway** 

**Recreation** 

**Robotics & Automation** 

# The GGB Advantage



#### **MAINTENANCE-FREE**

GGB bearings are self-lubricating, making them ideal for applications requiring long bearing life without continuous lubrication.



### LOW FRICTION, HIGH WEAR RESISTANCE

GGB bearings are self-lubricating, making them ideal for applications requiring long bearing life without continuous lubrication.



# NVH (NOISE, VIBRATION, HARSHNESS)

Plain bearings provide a smooth sliding motion between surfaces and their material properties and simple design reduce noise, vibration and harshness.



#### **LOWER SYSTEM COST**

A one-piece design offers space and weight reductions and thanks to the material compositions and self-lubricating properties, less maintenance is needed.



# REDUCED CO<sub>2</sub> FOOTPRINT

GGB's flexible and local production platforms assure timely deliveries and reduced CO<sub>2</sub> footprint.



#### PARTNER SUPPORT

GGB offers tribological, application and design support, and partners with our customers to provide the most efficient solutions.



# The Highest Standards in Fabrication

Our world-class manufacturing plants in the United States, Brazil, China, Germany, France and Slovakia are certified in quality and excellence according to ISO 9001, IATF 16949, ISO 14001 and ISO 45001. This allows us to access the industry's best practices while aligning our management system with global standards.

For a complete listing of our certifications, please visit our website:

www.ggbearings.com/en/certificates

# What is Tribology

# TRIBOLOGY IS THE SCIENCE OF FRICTION, WEAR, AND LUBRICATION

Tribology is the science of wear, friction and lubrication, and encompasses how interacting surfaces and other tribo-elements behave in relative motion in natural and artificial systems. This includes bearing design and lubrication.

### TRIBOLOGY SURROUNDS YOU

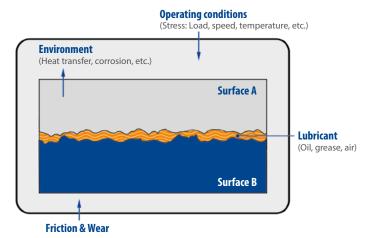
Tribology is everywhere where contacting surfaces are in relative movement to each other.



Tribology is the science of wear, friction and lubrication, and encompasses how interacting surfaces and other tribo-elements behave in relative motion in natural and artificial systems. This includes bearing design and lubrication.

### THE TRIBOLOGICAL SYSTEM

Tribology is a complex science, involving 2 surfaces in relative motion that are subject to constant mechanical, thermal and chemical interaction.



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# 1 Introduction

The purpose of this handbook is to provide comprehensive technical information on the characteristics of DP4® and DP4-B bearings.

The information given permits designers to establish the correct size of bearing required and the expected life and performance.

In addition, your local sales representative is available to assist you with your design. Complete information on the range of DP4® standard stock products is given together with details of other DP4® products.

GGB is continually refining and extending its experimental and theoretical knowledge and, therefore, when using this brochure it is always worth-while to contact GGB should additional information be required.

As it is impossible to cover all conditions of operation which arise in practice, customers are advised to carry out prototype testing wherever possible.

### 1.1 CHARACTERISTICS AND ADVANTAGES

The DP4® and DP4-B materials offer the following characteristics:

- Good frictional properties with negligible stick-slip
- High static and dynamic load capacity
- Suitable for rotating, oscillating, reciprocating and sliding movements
- Compact size and low weight
- Prefinished that requires no machining after assembly
- Possibility to burnish for reduced operating clearance
- No water absorption and therefore dimensionally stable
- Suitable for a wide operating temperature range from 200 to  $+280\,^{\circ}\text{C}$
- DP4-B with bronze backing for increased corrosion resistance
- Lead free in compliance with European RoHS 2002/95/EC, 2002/96/EC and EVL 2000/53/EC directives (see page 59)

In particular, depending on the dry running conditions, DP4® and DP4-B materials present the following performance advantages:

#### **DRY CONDITIONS**

- Good friction and wear performance under light duty conditions
- Particularly suitable for intermittent oscillating and reciprocating movements
- Maintenance free as no external lubrication required
- Seizure resistant.

#### **LUBRICATED CONDITIONS**

- Good wear and friction performance over a wide range of load, speed and temperature conditions
- High wear resistance in boundary operating conditions
- High resistance of bearing surface under fluid cavitation and flow erosion conditions
- Suitable for operation in diverse fluids (oil, fuel, solvents, refrigerants, water).

# 1 Introduction

# **1.2 APPLICATIONS**

Given the performance characteristics in both dry and lubricated operating conditions, DP4® and DP4-B bearing materials are extensively used in a wide range of automotive and industrial applications, such as:

#### **AUTOMOTIVE**

Braking systems, clutches, gearbox and transmissions, hinges - door bonnet and boot, convertible roof tops, pedal systems, pumps - axial, radial, gear and vane, seat mechanisms, steering systems, struts and shock absorbers, wiper systems.

#### **INDUSTRIAL**

Aerospace, agricultural, construction equipment, food and beverage, marine, material handling, office equipment, packaging equipment, pneumatic and hydraulic cylinders, railroad and tramways, textile machinery, valves.



# 2 Structure and Composition

#### DP4® / DP4-B

DP4° is a composite bearing material. It consists of a steel DP4° / bronze DP4-B backing to which is bonded a porous sinter bronze interlayer which is overlaid and impregnated with Polytetrafluoroethylene (PTFE) containing a mixture of inorganic fillers and special polymer fibres. The steel DP4° / bronze DP4-B backing provides mechanical strength and the bronze sinter layer provides a strong mechanical bond for the filled bearing lining.

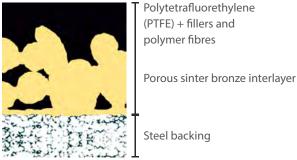
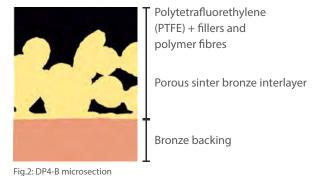


Fig.1: DP4 microsection



2.1 BASIC FORMS

#### STANDARD COMPONENTS

 $These\ products\ are\ manufactured\ to\ International,\ National\ or\ GGB\ standards.\ The\ following\ components\ are\ standard\ stock\ products:$ 



Fig.3: Standard stock products

#### **NON-STANDARD COMPONENTS**

These products are manufactured to customer's requirements and include for example:

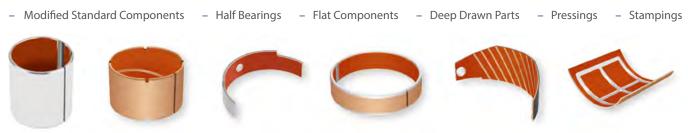


Fig.4: Non-standard components

# **3 Properties**

# 3.1 PHYSICAL AND MECHANICAL PROPERTIES

BEARING PROPERTIES		SYMBOL	UNIT	VAI DP4®	LUE DP4-B	COMMENTS	
PHYSICAL PROPERTIES	•						
Coefficient of linear thermal expansion	parallel to surface normal to service	$\alpha_1$ $\alpha_2$	10 <sup>-6</sup> /K	11 30	18 36		
Operating temperature		$T_{max}$ $T_{min}$	°C	+280 - 200	+280 - 200		
MECHANICAL PROPERTIES							
Compressive yield strength		$\sigma_{C}$	MPa	350	300	measured on disc Ø 25 mm x 2.45 mm thick	
Maximum load	static dynamic	P <sub>sta.max</sub> P <sub>dyn.max</sub>	MPa	250 140	140 140		

Table 1: Physical and mechanical properties of DP4 and DP4-B

CHEMICAL	%	°C	DP4®	DP4-B					
STRONG ACIDS									
Hydrochloric Acid	5	20	-	-					
Nitric Acid	5	20	-	-					
Sulfuric Acid	5	20	-	-					
WEAK ACIDS	WEAK ACIDS								
Acetic Acid	5	20	-	0					
Formic Acid	5	20	-	0					
BASES									
Ammonia	10	20	0	-					
Sodium Hydroxide	5	20	0	0					

CHEMICAL	°C	DP4®	DP4-B
SOLVENTS			
Acetone	20	+	+
Carbon Tetrachloride	20	+	+
LUBRICANTS AND FUELS			
Paraffin	20	+	+
Gasolene	20	+	+
Kerosene	20	+	+
Diesel Fuel	20	+	+
Mineral Oil	70	+	+
HFA-ISO46 High Water Fluid	70	+	+
HFC-Water-Glycol	70	+	+
HFD-Phosphate Ester	70	+	+
Water	20	0	+
Sea Water	20	-	0

Table 2: Chemical Resistance of DP4 and DP4-B

- + Satisfactory: Corrosion damage is unlikely to occur
- o Acceptable: Some corrosion damage may occur but this will not be sufficient to impair either the structural integrity or the tribological performance of the material
- Unsatisfactory: Corrosion damage will occur and is likely to affect either the structural integrity and/or the tribological performance of the material

#### **ELECTROCHEMICAL CORROSION**

DP4-B should not be used in conjunction with aluminium housings due to the risk of electrochemical corrosion in the presence of water or moisture.

### 3.3 FRICTIONAL PROPERTIES

DP4° bearings show negligible 'stick-slip' and provide smooth sliding between adjacent surfaces. The coefficient of friction of DP4° depends upon:

- The specific load P [MPa]
- The sliding speed U [m/s]
- The roughness of the mating running surface Ra [μm]

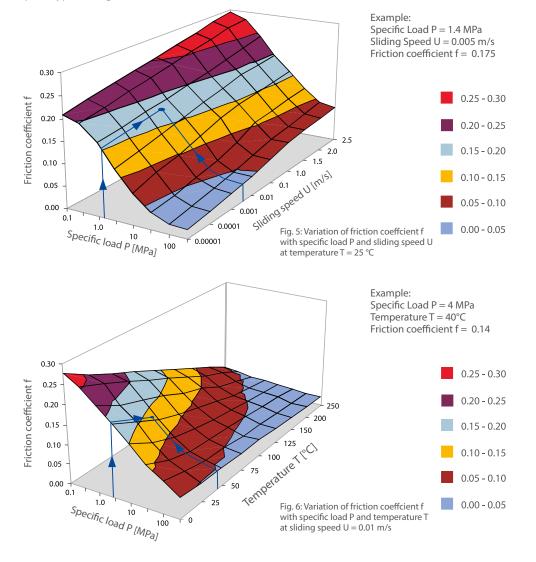
- The bearing temperature T [°C].

A typical relationship is shown in Fig. 5, which can be used as a guide to establish the actual friction under clean, dry conditions after running in. Exact values may vary by  $\pm 20\%$  depending on operating conditions. Before running in, the friction may be up to 50% higher.

After progressively longer periods of dwell under load (e.g. hours or days) the static coefficient of friction on the first movement may be between 1.5 and 3 times greater, particularly before running in.

#### **EFFECT OF TEMPERATURE FOR UNLUBRICATED APPLICATIONS**

The coefficient of friction of DP4° varies with temperature. Typical values are shown in Fig. 6 for temperatures up to 250 °C. Friction increases at bearing temperatures below 0 °C. Where frictional characteristics are critical to a design they should be established by prototype testing.



### 4.1 MCPHERSON STRUT APPLICATIONS

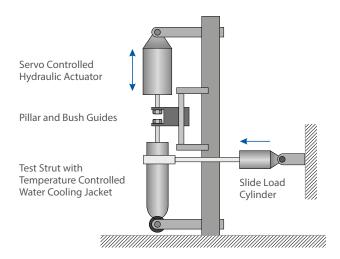
DP4° has been developed to provide improved wear, erosion resistance and reduced friction in McPherson strut piston rod guide bush applications under the most demanding of operating conditions.

In the following sections, the performance of DP4° is compared with that of the material used in the majority of this type of application.

#### **WEAR AND FRICTION PROPERTIES**

The wear and frictional performance of DP4° has been evaluated in the piston rod guide bush application of a McPherson strut shock absorber using the test rig shown in Fig. 7. The test conditions are designed to simulate the operational duty of the test strut in service and differ in detail according to the strut manufacturer. The test conditions used are given in Table 3 and Table 4.

#### MCPHERSON STRUT TEST RIG



Operating Cycle Sinusoidal Wear Test

Time

Fig. 7: Principle of the strut test rig

STRUT WEAR - TEST CONDITIONS						
Waveform	Sine					
Frequency	2.5 Hz					
Side load	890 N					
Test duration	100 hours					
Stroke	100 mm					
Mean diametral clearance	0.06 mm					
Lubricant	TEX 0358					
Foot valve temperature	70 °C					

Table 3: McPherson strut wear test conditions

STRUT FRICTION - TEST CONDITIONS						
Waveform	Sine					
Frequency	0.1 Hz					
Side load	600 N					
Stroke	70 mm					
Mean diametral clearance	0.06 mm					
Lubricant	TEX 0358					
Foot valve temperature	ambient					

Table 4: McPherson strut friction test conditions

The relative wear and frictional performance of DP4® tested under these conditions are shown in Figures 8 - 10. Actual results for the wear rate and friction are not quoted because these depend strongly on the actual test conditions and design of the strut under test. The relative performance plots shown thus provide the best indication as to the benefits offered by DP4® in this class of application.

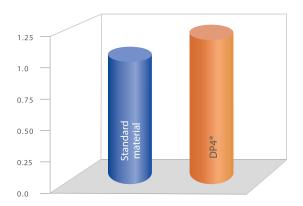


Fig. 8: Relative wear resistance

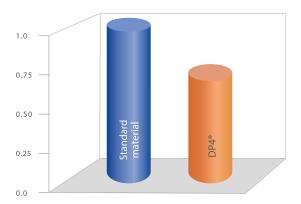


Fig. 10: Relative dynamic friction coefficient

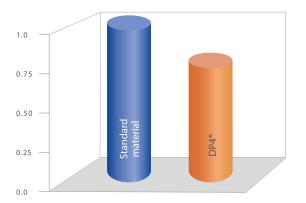


Fig. 9: Relative static friction coefficient

#### **CAVITATION EROSION RESISTANCE**

Under certain operating conditions, the PTFE lining of the McPherson strut piston rod guide bush can suffer erosion damage, due to cavitation and flow erosion effects from the oil film within the bearing. The test rig shown in Fig. 11 is designed to reproduce the cavitation erosion damage to the bearing lining of the test specimen. The test conditions are given in Table 5.

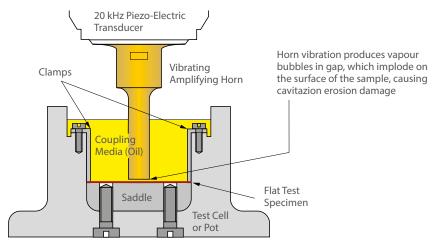


Fig. 11: Principle of the cavitation erosion test rig

CAVITATION EROSION - TEST CONDITIONS						
Amplitude	0.015 mm					
Frequency	20 kHz					
Separation	1 mm					
Test duration	30 minutes					
Lubricant	TEX 0358					
Temperature	ambient					

Table 5: Cavitation erosion test conditions

The relative resistance to cavitation damage of DP4 $^\circ$  as evaluated on this test rig is shown in Fig. 12.

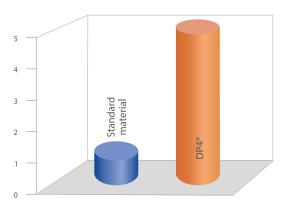


Fig. 12: Relative resistance to cavitation erosion

#### **FLOW EROSION RESISTANCE**

The test rig shown in Fig. 13 is designed to reproduce flow erosion damage to the bearing lining of the test specimen.

The test conditions are given in Table 6.

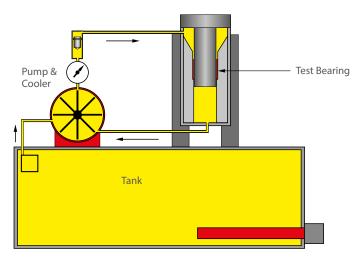


Fig. 13: Principle of the flow erosion test rig

FLOW EROSION - TEST CONDITIONS						
Bearing diameter	20 mm					
Bearing length	15 mm					
Diametral clearing	0.11 mm					
Pressure	13.8 MPa					
Flow rate	5 l/min					
Test duration	20 hours					
Shaft surface finish	0.15 μm ±0.05					
Temperature	ambient					

Table 6: Flow erosion test conditions

The relative resistance to flow erosion damage of DP4 $^\circ$  as evaluated on this test rig is shown in Fig. 14.

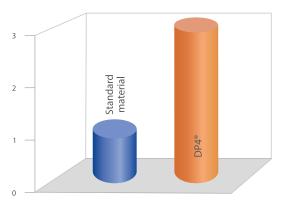


Fig. 14: Relative resistance to flow erosion

### 4.2 HYDRAULIC APPLICATIONS

DP4° also shows excellent wear and frictional performance in a wide range of oil lubricated hydraulic applications. The wear resistance of DP4° under steady load oil immersed boundary lubrication conditions has been evaluated using the test rig shown in Fig. 15. The test conditions are given in Table 7.

#### **GGB JUPITER TEST RIG**

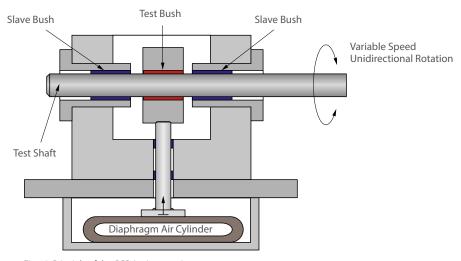


Fig. 15: Principle of the GGB Jupiter test rig

LUBRICATED WEAR - TEST CONDITIONS						
Bearing diameter	20 mm					
Bearing length	15 mm					
Mean diametral clearing	0.10 mm					
Speed	0.11 m/s					
Lubricant	ISO VG 46 hydraulic oil					

Table 7: Lubricated wear test conditions

The relative PU limits with boundary lubrication of DP4® and the material used in many high performance hydraulic pump applications as determined from these tests are shown in Fig. 16. The limiting PU depends upon the actual operating conditions and hence the relative performance only is given for guidance.

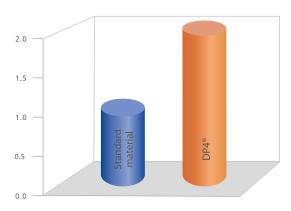


Fig. 16: Relative PU limits

### **4.3 DRY WEAR PERFORMANCE**

#### **DESGIN FACTORS**

The main parameters when determining the size or calculating the service life for a DP4® bearing are:

- Specific load limit P<sub>lim</sub>
- PU Factor
- Mating surface roughness R<sub>a</sub>

- Mating surface material
- Temperature T
- Other environmental factors e.g. housing design, dirt, lubricationtions.

The following calculation can be used to estimate the bearing service life of DP4® under dry running conditions.

#### **SPECIFIC LOAD P**

For the purpose of assessing bearing performance the specific load p is defined as the working load divided by the projected area of the bearing and is expressed in MPa.

#### CYLINDRICAL BUSH

(4.3.1) 
$$P = \frac{F}{D_i \cdot B}$$

#### THRUST WASHER

(4.3.2) [MPa] 
$$P = \frac{4F}{\pi \cdot (D_0^2 - D_1^2)}$$

#### **FLANGED BUSH (AXIAL LOADING)**

(4.3.3) 
$$P = \frac{F}{0,04 \cdot (D_{fl}^2 - D_l^2)}$$

#### **SLIDEWAY**

$$(4.3.4) \hspace{1cm} [MPa]$$
 
$$P = \frac{F}{L \cdot W}$$

### SPECIFIC LOAD LIMIT Plim

The maximum load which can be applied to a DP4® bearing can be expressed in terms of the Specific Load Limit, which depends on the type of the loading. It is highest under steady loads. Conditions of dynamic load or oscillating movement which produce fatigue stress in the bearing result in a reduction in the permissible Specific Load Limit.

In general the specific load on a DP4° bearing should not exceed the Specific Load Limits given in Table 8.

The values of Specific Load Limit specified in Table 8 assume good alignment between the bearing and mating surface (Fig. 35, page 33).

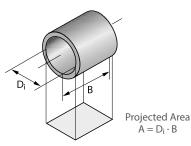


Fig. 17: Projected Area

#### MAXIMUM SPECIFIC LOAD Plim

TYPE OF LOADING / P <sub>lim</sub> [MPa]										
Steady load - rotating movement P <sub>lim</sub>	140									
Steady load - oscillating movement										
P <sub>lim</sub>	140	140	115	95	85	80	60	44	30	20
Number of movement cycles Q	1000	2000	4000	6000	8000	10 <sup>4</sup>	10 <sup>5</sup>	10 <sup>6</sup>	10 <sup>7</sup>	10 <sup>8</sup>
Dynamic load - rotating or oscillating movement										
P <sub>lim</sub>	60	60	50	46	42	40	30	22	15	10
Number of load cycles Q	1000	2000	4000	6000	8000	10 <sup>4</sup>	10 <sup>5</sup>	10 <sup>6</sup>	10 <sup>7</sup>	108

Table 8: Maximum specific load Plim

Permanent deformation of the DP4° bearing lining may occur for specific loads above 140 MPa unless with slow intermittent movements. Under these conditions, it is recommended to contact GGB for further information.

The permissible maximum load on a thrust washer is higher than that on the flange of a flanged bush, and under conditions of high axial loads a thrust washer should be specified.

#### **SLIDING SPEED U**

Speeds in excess of 2.5 m/s sometimes lead to overheating, and a running in procedure may be beneficial. This could consist of a series of short runs progressively increasing in duration from an initial run of a few seconds.

#### **CONTINUOUS ROTATION**

#### CYLINDRICAL BUSH

$$(4.3.5) \hspace{1cm} [m/s]$$
 
$$U = \frac{D_i \cdot \pi \cdot N}{60 \cdot 10^3}$$

#### **THRUST WASHER**

(4.3.6) 
$$U = \frac{\frac{D_0 + D_i}{2} \cdot \pi \cdot N}{60 \cdot 10^3}$$
 [m/s]

#### **OSCILLATING MOVEMENT**

#### CYLINDRICAL BUSH

(4.3.7) 
$$U = \frac{D_i \cdot \pi}{60 \cdot 10^3} \cdot \frac{4\phi \cdot N_{OSZ}}{360}$$

#### **THRUST WASHER**

(4.3.8) 
$$U = \frac{\frac{D_o + D_i}{2} \cdot \pi}{60 \cdot 10^3} \cdot \frac{4\phi \cdot N_{OSZ}}{360}$$

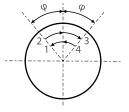


Fig. 18: Oscillating cycle φ

#### **PU FACTOR**

The useful operating life of a DP4® bearing is governed by the PU factor, the product of the specific load P [MPa] and the sliding speed U [m/s].

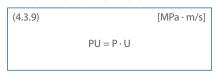
For thrust washers and flanged bush thrust faces the rubbing velocity at the mean diameter is used.

PU factors up to 1.0 MPa x m/s can be accommodated for short periods, whilst for continuous rating, PU factors up to 0.5 MPa x m/s can be used, depending upon the operating life required.

	DU	UNIT
Р	140	MPa
U	2.5	m/s
PU continuous	0.5	MPa·m/s
PU intermittent	1.0	MPa · m/s

Table 9: Typical data P, U and PU

#### **CALCULATION OF PU FACTOR**



#### **APPLICATON FACTORS**

The following factors influence the bearing performance of DP4® and must be considered in calculating the required dimension or estimating the bearing life for a particular application.

#### **TEMPERATURE**

The useful life of a DP4® bearing depends upon the operating temperature.

Under dry running conditions frictional heat is generated at the rubbing surface of the bearing dependent on the PU condition. For a given PU factor the operating temperature of the bearing depends upon the temperature of the surrounding environment, the heat dissipation properties of the housing and the mating surface. Intermittent operation affects the heat dissipation from the assembly and hence the operating temperature of the bearing.

The effect of temperature on the operating life of DP4® bearings is indicated by the factor a<sub>T</sub> shown in Table 10.

MODE OF OPERATION	NATURE OF HOUSING	TEMPERATURE OF BEARING ENVIRONMENT T <sub>amb</sub> [°C] AND TEMPERATURE APPLICATION FACTOR a <sub>T</sub>					
		25	60	100	150	200	280
Dry continuous operation	Average heat dissipating qualities	1.0	0.8	0.6	0.4	0.2	0.1
Dry continuous operation	Light pressings or isolated housing with poor heat dissipating qualities	0.5	0.4	0.3	0.2	0.1	-
Dry continuous operation	Non-metallic housings with bad heat dissipating qualities	0.3	0.3	0.2	0.1	-	-
<b>Dry intermittent operation</b> (duration less than 2 min, followed by a longer dwell period)	Average heat dissipating qualities	2.0	1.6	1.2	0.8	0.4	0.2

Table 10: Temperature application factor  $a_T$ 

#### **MATING SURFACE**

The effect of the mating surface material type on the operating life of DP4 $^{\circ}$  bearings is indicated by the mating surface factor  $a_{M}$  and the life correction constant aL shown in Table 11.

MATERIAL	a <sub>M</sub>	аL
STEEL AND CAST IRON		
Carbon Steel	1	400
Carbon Manganese Steel	1	400
Alloy Steel	1	400
Case Hardened Steel	1	400
Nitrided Steel	1	400
Salt bath nitrocarburised	1	400
Stainless Steel (7-10% Ni, 17-20% Cr)	2	400
Cast Iron (0.3 $\pm$ 0.1 $\mu$ m R <sub>a</sub> )	1	400

Table 11: Mating surface factor a<sub>M</sub> and life correction constant a<sub>L</sub>

#### NOTE:

The factor values given assume a mating surface finish of  $R_a = 0.4 \pm 0.1 \; \mu m.$ 

- A ground surface is preferred to fine turned
- Surfaces should be cleaned of abrasive particles after polishing
- Cast iron surfaces should be ground to Ra =  $0.3 \pm 0.1 \mu m$
- The grinding cut should be in the same direction as the bearing motion relative to the shaft

#### **BEARING SIZE**

The running clearance of a DP4® bearing increases with bearing diameter resulting in a proportionally smaller contact area between the shaft and bearing. This reduction in contact area has the effect of increasing the actual unit load and hence PU factor. The bearing size factor (Fig. 20) is used in the design calculations to allow for this effect.



Fig. 19: Contact area between bearing and shaft

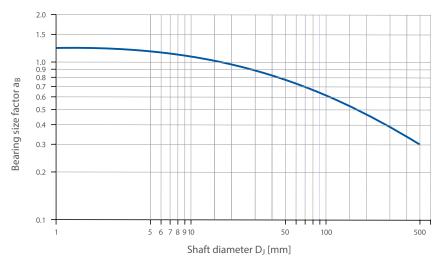


Fig. 20: Bearing size factor a<sub>B</sub>

#### **BORE BURNISHING**

Burnishing or machining the bore of a DP4° bearing results in a reduction in the wear performance. The application factor a<sub>C</sub> given in table 12 is used in the design calculations to allow for this effect. Machining DP4° is not recommended.

DEGREE OF SIZING		APPLICATION FACTOR a <sub>C</sub>
BURNISHING	0.025 mm	0.8
Excess of burnishing tool diameter over	0.038 mm	0.6
mean bore size	0.050 mm	0.3

Table 12: Bore burnishing or machining application factor  $a_{C}$ 

#### **TYPE OF LOAD**

The type of load is considered in formula (4.4.9) page 23 and (4.4.10) page 23.

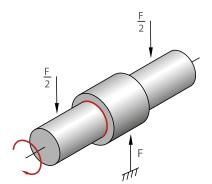


Fig. 21: Steady load, bush stationary, shaft rotating

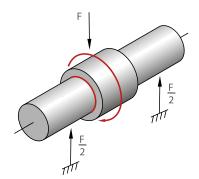


Fig. 22: Rotating load, shaft stationary, bush rotating

### 4.4CALCULATION OF BEARING SERVICE LIFE

Where the size of a bearing is governed largely by the space available the following calculation can be used to determine whether its useful life will satisfy the requirements. If the calculated life is inadequate, a larger bearing should be considered.

#### **SPECIFIC LOAD P**

#### **BUSHES**

$$(4.4.1) \label{eq:paper_part} P = \frac{F}{D_i \cdot B}$$

#### **FLANGED BUSHES**

(4.4.2) 
$$P = \frac{F}{0.04 \cdot (D_{fl^2} - D_{i^2})} \label{eq:paper}$$

#### **THRUST WASHERS**

$$P = \frac{4F}{P \cdot (D_o{}^2 - D_i{}^2)} \label{eq:paper}$$

#### HIGH LOAD FACTOR aE

(4.4.4) 
$$a_{E} = \frac{P_{lim} - P}{P_{lim}}$$

If a<sub>E</sub> is negative then the bearing is overloaded. Increase the bearing diameter and/or length.

#### **BUSHES**

$$(4.4.5) \qquad \qquad [MPa \cdot m/s]$$
 
$$PU = \frac{5.25 \cdot 10^{-5} F \cdot N}{a_E \cdot B \cdot a_T \cdot a_M \cdot a_B}$$

#### **FLANGED BUSHES**

$$(4.4.6) \qquad [MPa \cdot m/s]$$
 
$$PU = \frac{6.5 \cdot 10^{-4}F \cdot N}{a_E \cdot (D_{fl} - D_{i}) \cdot a_T \cdot a_M \cdot a_B}$$

#### **THRUST WASHERS**

$$PU = \frac{3.34 \cdot 10^{-5} F \cdot N}{a_E \cdot (D_o - D_i) \cdot a_T \cdot a_M \cdot a_B}$$

For oscillating movement, calculate the average rotational speed.

$$(4.4.8) \hspace{1cm} [1/min]$$
 
$$N = \frac{4\phi \cdot N_{osz}}{360}$$

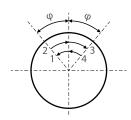


Fig. 23: Oscillating cycle  $\boldsymbol{\phi}$ 



#### ESTIMATION OF BEARING LIFE LH

#### **BUSHES (STEADY LOAD)**

(4.4.9) [h] 
$$L_{H} = \frac{265}{PU} - a_{L}$$

#### **BUSHES (ROTATING LOAD)**

(4.4.10) [h] 
$$L_{H} = \frac{530}{PU} - a_{L}$$

#### **FLANGED BUSHES (AXIAL LOAD)**

(4.4.11) [h] 
$$L_{H} = \frac{175}{PU} - a_{L}$$

#### **THRUST WASHERS**

(4.4.12) [h] 
$$L_{H} = \frac{175}{PU} - a_{L}$$

#### **BORE BURNISHING**

If the DP4 $^{\circ}$  bush is bore burnished then this must be allowed for in estimating the bearing life by the application factor  $a_{C}$  (Table 12, page 21).

#### **ESTIMATED BEARING LIFE**

(4.4.13) 
$$L_{H} = L_{H} \cdot a_{C}$$

#### FOR OSCILLATING MOVEMENTS OR DYNAMIC LOADS

(4.4.14) [cycles] 
$$Z_T = L_H \cdot N_{OSC} \cdot 60 \label{eq:ZT}$$
 for oscillating movements

(4.4.15) [cycles]  $Z_T = L_H \cdot C \cdot 60 \label{eq:ZT}$  for dynamic loads

Calculate estimated number of cycles Z<sub>T</sub>

Check that  $Z_T$  is less than total number of cycles Q for the operating specific load P (Table 8, page 18)

If  $Z_T < Q$ ,  $L_H$  will be limited by wear after  $Z_T$  cycles.

If  $Z_T > Q$ ,  $L_H$  will be limited by fatigue after  $Z_T$  cycles.

#### **SLIDEWAYS**

### SPECIFIC LOAD FACTOR

(4.4.16) 
$$a_{E1} = A - \frac{F}{P_{lim}}$$

If negative the bearing is overloaded and the bearing area should be increased.

# SPEED TEMPERATURE AND MATERIAL APPLICATION FACTORS

(4.4.17) 
$$a_{E2} = \frac{280 \cdot a_T \cdot a_M}{F \cdot U}$$

#### **RELATIVE CONTACT AREA FACTOR**

(4.4.18) 
$$a_{E3} = \frac{A}{A_{M}}$$

#### **ESTIMATED BEARING LIFE**

(4.4.19) 
$$L_{H} = a_{E1} \cdot a_{E2} \cdot a_{E3} - a_{L}$$

#### NOTE:

Estimated bearing lives greater than 4000 h are subject to error due to inaccuracies in the extrapolation of test data.

# **4.5 WORKED EXAMPLES**

### CYLINDRICAL BUSH

Given:				
Load Details	Steady Load	Inside Diameter D <sub>i</sub>	40 mm	
	Continuous Rotation	Length B	30 mm	
Shaft	Steel	Bearing Load F	5.000 N	
	Unlubricated at 25°C	Rotational Speed N	25 · 1/min	

Calculation Constants and Application Factors		
Specific Load Limit P <sub>lim</sub>	140 MPa	(Table 8, page 18)
Temperature Application Factor a <sub>T</sub>	1.0	(Table 10, page 19)
Material Application Factor a <sub>M</sub>	1.0	(Table 11, page 20)
Bearing Size Factor a <sub>B</sub>	0.85	(Fig. 20, page 21)
Life Correction Constant a <sub>L</sub>	400	(Table 11, page 20)

Calculation	Ref	Value
Specific Load P [MPa]	(4.4.1) Page 22	$P = \frac{F}{D_i \cdot B} = \frac{5.000}{40 \cdot 30} = 4.17$
Sliding Speed U [m/s]	(4.3.5) Page 18	$U = \frac{D_i \cdot \pi \cdot N}{60 \cdot 10^3} = \frac{40 \cdot 3.14 \cdot 25}{60 \cdot 10^3} = 0.052$
High Load Factor a <sub>E</sub> [-] must be > 0	(4.4.4) Page 22	$a_E = \frac{P_{lim} - P}{P_{lim}} = \frac{140 - 4.17}{140} = 0.97$
Modified PU Factor [MPa · m/s]	(4.4.5) Page 22	$PU = \frac{5.25 \cdot 10^{-5} F \cdot N}{a_E \cdot B \cdot a_T \cdot a_M \cdot a_B} = 0.27$
Life L <sub>H</sub> [h]	(4.4.9) Page 23	$L_{H} = \frac{265}{PU} - a_{L} = \frac{265}{0.27} - 400 = 581$

### **FLANGED BUSH**

Given:			
Load Details	Axial Load Continuous Rotation	Flange Outside Ø D <sub>fl</sub> Inside Diameter D <sub>i</sub>	
Shaft	Steel Unlubricated at 25°C	Bearing Load F Rotational Speed N	250 N 5 · 1/min

Calculation Constants and Application Factors		
Specific Load Limit P <sub>lim</sub>	140 MPa	(Table 8, page 18)
Temperature Application Factor a <sub>T</sub>	1.0	(Table 10, page 19)
Material Application Factor a <sub>M</sub>	1.0	(Table 11, page 20)
Bearing Size Factor a <sub>B</sub>	1.0	(Fig. 20, page 21)
Life Correction Constant a <sub>L</sub>	400	(Table 11, page 20)

Calculation	Ref	Value
Specific Load P [N/mm²]	(4.4.2) Page 22	$P = \frac{250}{0.04 \cdot (23^2 - 15^2)} = 20.55$
Sliding Speed U [m/s]	(4.3.6) Page 18	$U = \frac{\frac{23+15}{2} \cdot 3,14 \cdot 5}{60 \cdot 10^3} = 0.005$
High Load Factor a <sub>E</sub> [-] must be > 0	(4.4.4) Page 22	$a_E = \frac{P_{lim} - P}{P_{lim}} = \frac{140 - 20.55}{140} = 0.835$
Modified PU Factor [N/mm²·m/s]	(4.4.6) Page 22	$PU = \frac{6.5 \cdot 10^{-4} \cdot 250 \cdot 5}{0.853 \cdot (23 - 15) \cdot 1 \cdot 1 \cdot 1} = 0,119$
Life L <sub>H</sub> [h]	(3.8.11) Page 21	$L_{H} = \frac{175}{PU} - a_{L} = \frac{175}{0.119} - 400 = 1071$

### **THRUST WASHER**

Given:			
Load Details	Axial Load	Outside Diameter Do	62 mm
	Continuous Rotation	Inside Diameter Di	38 mm
Shaft	Steel	Bearing Load F	6.500 N
	Unlubricated at 25°C	Rotational Speed N	10 · 1/min

Calculation Constants and Application Factors		
Specific Load Limit P <sub>lim</sub>	140 MPa	(Table 8, page 18)
Temperature Application Factor a <sub>T</sub>	1.0	(Table 10, page 19)
Material Application Factor a <sub>M</sub>	1.0	(Table 11, page 20)
Bearing Size Factor a <sub>B</sub>	0.85	(Fig. 20, page 21)
Life Correction Constant a <sub>L</sub>	400	(Table 11, page 20)

		·
Calculation	Ref	Value
Specific Load P [N/mm²]	(4.4.3) Page 22	$P = \frac{4 \cdot 6.500}{3.14 \cdot (62^2 - 38^2)} = 3.45$
Sliding Speed U [m/s]	(4.3.6) Page 18	$U = \frac{\frac{62 + 38}{2} \cdot 3,14 \cdot 10}{60 \cdot 10^3} = 0.026$
High Load Factor a <sub>E</sub> [-] must be > 0	(4.4.4) Page 22	$a_E = \frac{P_{lim} - P}{P_{lim}} = \frac{140 - 3.45}{140} = 0.975$
Modified PU Factor [MPa·m/s]	(4.4.7) Page 22	$PU = \frac{3.34 \cdot 10^{-5} \cdot 6.500 \cdot 10}{0.975 \cdot (62 - 38) \cdot 1 \cdot 1 \cdot 0.85} = 0.11$
Life L <sub>H</sub> [h]	(4.4.12) Page 23	$L_{H} = \frac{175}{PU} - a_{L} = \frac{175}{0.11} - 400 = 1191$

# **5 Lubrication**

DP4° provides excellent performance in lubricated applications. The following sections describe the basics of lubrication and provide guidance on the application of DP4° in such environments.

### **5.1 LUBRICANTS**

DP4® can be used with most fluids including:

- water
- lubricating oils
- engine oil
- turbine oil
- hydraulic fluid
- solvent
- refrigerants

In general, the fluid will be acceptable if it does not chemically attack the PTFE/lead overlay or the porous

bronze interlayer. Where there is doubt about the suitability of a fluid, a simple test is to submerge a sample of DP4° material in the fluid for two to or three weeks at 15-20 °C above the operating temperature.

The following will usually indicate that the fluid is not suitable for use with DP4°:

- A significant change in the thickness of the DP4<sup>®</sup> material,
- a visible change in the bearing surface other than some discolouration or staining,
- a visible change in the microstructure of the bronze interlayer.

### **5.2 TRIBOLOGY**

There are three modes of lubricated bearing operation which relate to the thickness of the developed lubricant film between the bearing and the mating surface.

- Hydrodynamic lubrication
- Mixed film lubrication
- Boundary lubrication

These three modes of operation depend upon:

- Bearing dimensions
- Clearance
- Load
- Speed
- Lubricant Viscosity
- Lubricant Flow

#### **HYDRODYNAMIC LUBRICATION**

#### **CHARACTERISED BY:**

- Complete separation of the shaft from the bearing by the lubricant film
- Very low friction and no wear of the bearing or shaft since there is no contact.
- Coefficients of friction of 0.001 to 0.01

Hydrodynamic conditions occur when:

$$(5.2.1) \label{eq:paper_problem} [MPa]$$
 
$$P \leq \frac{U \cdot \eta}{7.5} \cdot \frac{B}{D_i}$$

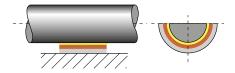


Fig. 24: Hydrodynamic lubrication

# **5 Lubrication**

#### MIXED FILM LUBRICATION

#### **CHARACTERISED BY:**

- Combination of hydrodynamic and boundary lubrication.
- Part of the load is carried by localised areas of self pressurised lubricant and the remainder supported by boundary lubrication.
- Friction and wear depend upon the degree of hydrodynamic support developed.
- DP4® provides low friction and high wear resistance to support the boundary lubricated element of the load.

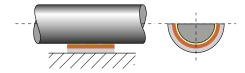


Fig. 25: Mixed film lubrication

#### **BOUNDARY LUBRICATION**

#### **CHARACTERISED BY:**

- Rubbing of the shaft against the bearing with virtually no lubricant separating the two surfaces.
- Bearing material selection is critical to performance
- Shaft wear is likely due to contact between bearing and shaft.
- The excellent self lubricating properties of DP4® material minimises wear under these conditions.
- The dynamic coefficient of friction with DP4° is typically 0.05 to 0.3 under boundary lubrication conditions.
- The static coefficient of friction with DP4® is typically slightly above the dynamic coefficient of friction under boundary lubrication conditions.

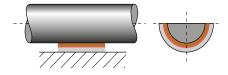


Fig. 26: Boundary lubrication

### **5.3 CHARACTERISTICS OF LUBRICATED DP4® BEARINGS**

DP4° is particularly effective in the most demanding of lubricated applications where full hydrodynamic operation cannot be maintained, for example:

#### - High load conditions

In highly loaded applications operating under boundary or mixed film conditions DP4° shows excellent wear resistance and low friction.

#### Start up and shut down under load

With insufficient speed to generate a hydrodynamic film the bearing will operate under boundary or mixed film conditions. DP4° minimises wear and requires less start up torque than conventional metallic bearings.

#### - Sparse lubrication

Many applications require the bearing to operate with less than the ideal lubricant supply, typically with splash or mist lubrication only. DP4° requires significantly less lubricant than conventional metallic bearings.

#### Non lubricating fluids

DP4® operates satisfactorily in low viscosity and non lubricating fluids such as water and some process fluids.

#### NOTE THE FOLLOWING HOWEVER:

If a DP4® bearing is required to run dry after running in water

under non hydrodynamic conditions then the wear resistance will be substantially reduced due to an increased amount of bedding in wear.

Fig. 27, page 28 shows the three lubrication regimes discussed above plotted on a graph of sliding speed vs the ratio of specific load to lubricant viscosity.

#### Using the formula in Section 4:

- Calculate the specific load P,
- calculate the shaft surface speed U.

# Using the viscosity temperature relationships presented in Table 13:

- Determine the viscosity in centipoise of the lubricant.

#### Note:

Viscosity is a function of operating temperature. If the operating temperature of the fluid is unknown, a provisional temperature of 25 °C above ambient can be used.

VISCOSITY ŋ [cP]															
TEMPERATURE [°C]	0	10	20	30	40	50	60	70	80	90	100	110	120	130	140
Lubricant															
ISO VG 32	310	146	77	44	27	18	13	9.3	7.0	5.5	4.4	3.6	3.0	2.5	2.2
ISO VG 46	570	247	121	67	40	25	17	12	9.0	6.9	5.4	4.4	3.6	3.0	2.6
ISO VG 68	940	395	190	102	59	37	24	17	12	9.3	7.2	5.8	4.7	3.9	3.3
ISO VG 100	2110	780	335	164	89	52	33	22	15	11.3	8.6	6.7	5.3	4.3	3.6
ISO VG 150	3600	1290	540	255	134	77	48	31	21	15	11	8.8	7.0	5.6	4.6
Diesel oil	4.6	4.0	3.4	3.0	2.6	2.3	2.0	1.7	1.4	1.1	0.95				
Petrol	0.6	0.56	0.52	0.48	0.44	0.40	0.36	0.33	0.31						
Kerosene	2.0	1.7	1.5	1.3	1.1	0.95	0.85	0.75	0.65	0.60	0.55				
Water	1.79	1.30	1.0	0.84	0.69	0.55	0.48	0.41	0.34	0.32	0.28				

Table 13: Dynamic viscosity

# **5 Lubrication**

### 5.4 DESIGN GUIDANCE

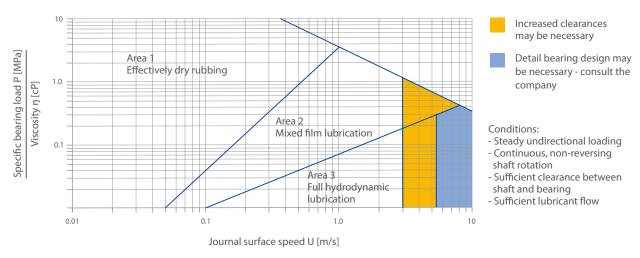


Fig. 27: Design guide for lubricated application

#### **EXPLANATION TO FIGURE 27**

#### AREA 1

The bearing will operate with boundary lubrication and PU factor will be the major determinant of bearing life. The DP4° bearing performance can be calculated using the method given in section 4, although the result will probably underestimate the bearing life

#### AREA 2

The bearing will operate with mixed film lubrication and the pU factor is no longer a significant parameter in determining the bearing life. The DP4® bearing performance will depend upon the nature of the fluid and the actual service conditions.

#### **AREA 3**

The bearing will operate with hydrodynamic lubrication. The bearing wear will be determined only by the cleanliness of the lubricant and the frequency of start up and shut down.

#### AREA 4

These are the most demanding operating conditions. The bearing is operated under either high speed or high bearing load to viscosity ratio, or a combination of both. These conditions may cause:

- excessive operating temperature and/or
- high wear rate.

The bearing performance may be improved by adding one or more grooves to the bearing and a shaft surface finish  $<\!0.05~\mu m~R_a.$ 

### 5.5 CLEARANCES FOR LUBRICATED OPERATION

The recommended shaft and housing diameters given for standard DP4® bushes will provide sufficient clearance for applications operating with boundary lubrication.

For bearings operating with mixed film or hydrodynamic lubrication it may be necessary to improve the fluid flow through the bearing by reducing the recommended shaft diameter by approximately 0.1 %, particularly when the shaft surface speed exceeds 2.5 m/s.

#### 5.6 GROOVING FOR LUBRICATED OPERATION

In demanding applications an axial oil groove will improve the performance of DP4°. Figure 28 shows the recommended form and location of a single groove with respect to the applied load and the bearing split. GGB can manufacture special DP4° bearings with embossed or milled grooves on request.

### 5.7 MATING SURFACE FINISH FOR LUBRICATED OPERATION

- $R_a \le 0.4 \pm 0.1 \,\mu m$  boundary lubrication
- $R_a = 0.1 0.2 \mu m$  mixed film or hydrodynamic conditions
- $R_a \le 0.05 \mu m$  for the most demanding operating conditions

### **5.8 GREASE LUBRICATION**

DP4° is not generally recommended for use with grease lubrication. In particular the following must be avoided:

- Dynamic loads which can result in erosion of the PTFE/lead bearing surface.
- Greases with EP additives or fillers such as graphite or MoS<sub>2</sub> which can cause rapid wear of DP4°.

Under grease lubrication, improved performance can be obtained by the use of other GGB metal polymer bearing materials, for example, DX®, DX®10, DS, HI-EX®.

Please contact your local sales representative or consult: https://www.ggbearings.com for more details.

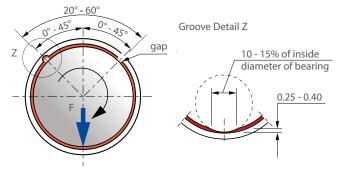


Fig. 28: Location of oil holes and grooves

# **6 Bearing Assembly**

#### **DIMENSIONS AND TOLERANCES**

DP4® bushes are prefinished and excluding very exceptional circumstances, must not be broached, machined or otherwise modified in the bore. It is essential that the correct running clearance is used and that both the diameter of the shaft and the bore of the housing are finished to the limits given in the tables. Under dry running conditions any increase in the clearances given will result in a proportional reduction in performance.

If the bearing housing is unusually flexible the bush will not close in by the calculated amount and the running clearance will be more than the optimum. In these circumstances the housing should be bored slightly undersize or the journal diameter increased, the correct size being determined by experiment.

Where free running is essential, or where light loads (less than 0.1 MPa) prevail and the available torque is low, increased clearance is required and it is recommended that the shaft size quoted in the table be reduced by 0.025 mm.

#### 6.1 ALLOWANCE FOR THERMAL EXPANSION

For operation in high temperature environments the clearance should be increased by the amounts indicated by figure 29 to compensate for the inward thermal expansion of the bearing lining.

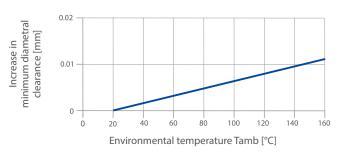


Fig. 29: Increase in diametral clearance

If the housing is non-ferrous then the bore should be reduced by the amounts given in Table 14, in order to give an increased interference fit to the bush, with a similar reduction in the journal diameter additional to that indicated by figure 29.

HOUSING MATERIAL	REDUCTION IN HOUSING DIAMETER PER 100°C RISE	REDUCTION IN SHAFT DIAMETER PER100°C RISE
Aluminium alloys	0.1 %	0.1 % + values from Fig.29
Copper base alloys	0.05 %	0.05 % + values from Fig.29
Steel and cast iron	-	values from Fig.29
Zinc base alloys	0.15 %	0.15 % + values from Fig.29

Table 14: Allowance for high temperature

### **6.2 TOLERANCES FOR MINIMUM CLEARANCE**

Where it is required to keep the variation of assembled clearance to a minimum, closer tolerances can be specified towards the upper end of the journal tolerance and the lower end of the housing tolerance. If housings to H6 tolerance are used, then the journals should be finished to the following limits. The sizes in Table 16 give the following nominal clearance range.

D <sub>i</sub>	D <sub>J</sub>
> 5 mm < 25 mm	-0.019 to -0.029
> 25 mm < 50 mm	-0.021 to -0.035

D <sub>i</sub>	Dj
10 mm	0.009 to 0.080
50 mm	0.011 to 0.134

Table 16: Clearance vs bearing diameter

#### **BURNISHING**

The burnishing or fine boring of the bore of an assembled DP4° bush in order to achieve a smaller clearance tolerance is only permissible if a substantial reduction in performance is acceptable. Fig. 24 shows a recommended burnishing tool for the sizing of DP4° bushes.

The coining section of the burnishing tool should be case hardened (case depth 0.6 - 1.2 mm, HRC 60±2) and polished with diamond paste (RZ  $\approx 1~\mu m$ ). A TiN type surface treatment increases the wear resistance of the burnishing tool and when absent gives a visual indication of burnishing tool wear.

Note: Ball burnishing of DP4® bushes is not recommended.

The values given in Table 17 indicate the dimensions of the burnishing tool required to give specific increases in the bearing bore diameter.

Exact values must be determined by test.

The reduction in bearing performance as a result of burnishing is allowed for in the bearing life calculation by the application factor aC (Table 12, page 21). The impact of burnishing on the bearing and assembly should be validated by trials.

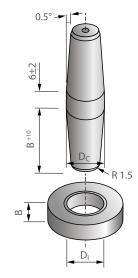


Fig. 30: Burnishing tool

ASSEMBLED BUSH INSIDE Ø	REQUIRED BUSH INSIDE Ø	REQUIRED BURNISHING TOOL Ø DC
D <sub>i,a</sub>	D <sub>i,a</sub> + 0.025	D <sub>i,a</sub> + 0.06
D <sub>i,a</sub>	D <sub>i,a</sub> + 0.038	$D_{i,a} + 0.08$
D <sub>i,a</sub>	D <sub>i,a</sub> + 0.050	$D_{i,a} + 0.1$

Table 17:Burnishing tool tolerances

# **6.3 COUNTERFACE DESIGN**

The suitability of mating surface materials and recommendations of mating surface finish for use with DP4° are discussed in detail on page 20.

DP4° is normally used in conjunction with ferrous journals and thrust faces, but in damp or corrosive surroundings, particularly without the protection of oil or grease, stainless steel, hard chromium plated mild steel, or hard anodised aluminium is recommended. When plated mating surfaces are specified the plating should possess adequate strength and adhesion, particularly if the bearing is to operate with high fluctuating loads.

The shaft or thrust collar used in conjunction with the DP4® bush or thrust washer must extend beyond the bearing surface in order to avoid cutting into it. The mating surface must also be free from grooves or flats, the end of the shaft should be given a lead-in chamfer and all sharp edges or projections which may damage the soft overlay of the DP4® must be removed.

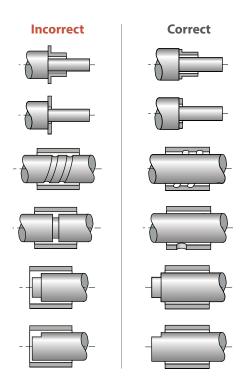


Fig. 31: Counterface Design

# **6 Bearing Assembly**

# **6.4 INSTALLATION**

#### FITTING OF CYLINDRICAL BUSHES

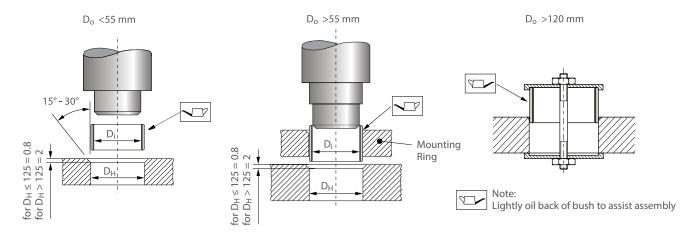
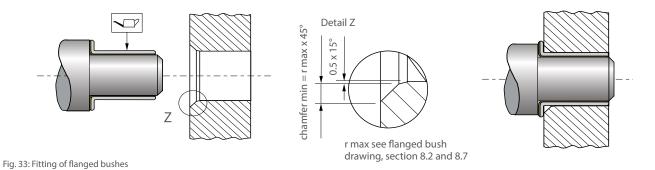


Fig. 32: Fitting of cylindrical bushes

#### FITTING OF FLANGED BUSHES



**INSERTION FORCES** 

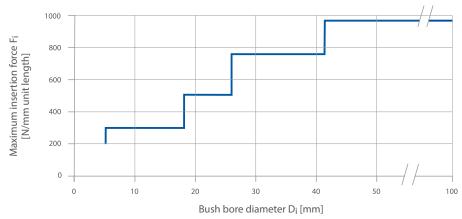


Fig. 34: Maximum Insertion Force Fi

#### ALIGNMENT

Accurate alignment is an important consideration for all bearing assemblies, but is particularly so for dry bearings because there is no lubricant to spread the load. With DP4® bearings misalignment over the length of a bush (or pair of bushes), or over the diameter of a thrust washer should not exceed 0.020 mm as illustrated in Fig. 35.

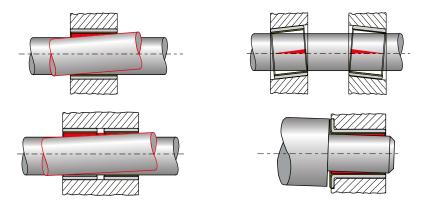


Fig. 35: Alignment

#### **SEALING**

While DP4° can tolerate the ingress of some contaminant materials into the bearing without loss of performance, where there is the possibility of highly abrasive material entering the bearing, a suitable sealing arrangement, as illustrated in Fig. 36 should be provided.

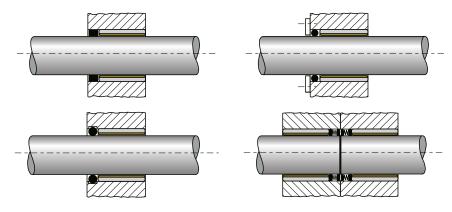


Fig. 36: Recommended sealing arrangements

### **6.5 AXIAL LOCATION**

Where axial location is necessary, it is advisable to fit DP4® thrust washers in conjunction with DP4® bushes, even when the axial loads are low.

#### **FITTING OF THRUST WASHERS**

DP4® thrust washers should be located in a recess as shown in Fig. 37. For the recess diameter the tolerance class [D10] is recommended. The recess depth is given in the product tables, page 44 and following. If a recess is not possible one of the following methods may be used:

Two dowel pins
 Two screws
 Adhesive
 Soldering (temperature < 320 °C).</li>

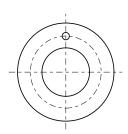
# **6 Bearing Assembly**

#### **IMPORTANT NOTE**

- Ensure the washer ID does not touch the shaft after assembly
- Ensure that the washer is mounted with the steel backing to the housing
- Dowels pins should be recessed 0.25 mm below the bearing surface
- Screws should be countersunk 0.25 mm below the bearing surface

- DP4® must not be heated above 320 °C
- Contact adhesive manufacturers for guidance selection of suitable adhesive
- Protect the bearing surface to prevent contact with adhesive





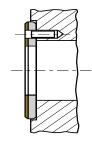


Fig. 37: Installation of Thrust Washer

#### **GROOVES FOR WEAR DEBRIS REMOVAL**

Tests with thrust washers have demonstrated that for optimum dry wear performance at specific loads in excess of 35 MPa, four wear debris removal grooves should be machined in the bearing surface as shown in Fig. 38.

Grooves in bushes have not been found to be beneficial in this respect.

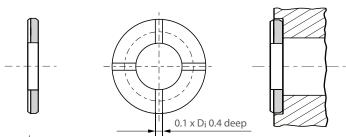
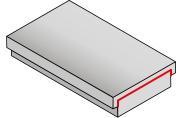


Fig. 38: Debris removal grooves

#### **SLIDEWAYS**

DP4® strip material for use as slideway bearings should be installed using one of the following methods:

- Countersunk screws - Adhesives - Mechanical location as shown in Fig. 39



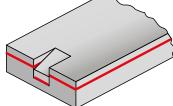


Fig. 39: Mechanical location of DU slideplates

# 7 Modification

### 7.1 CUTTING AND MACHINING

The modification of DP4® bearing components requires no special procedures. In general it is more satisfactory to perform machining or drilling operations from the PTFE side in order to avoid burrs. When cutting is done from the steel side, the minimum cutting pressure should be used and care taken to ensure that any steel or bronze particles protruding into the remaining bearing material, and all burrs, are removed.

#### **DRILLING OIL HOLES**

Bushes should be adequately supported during the drilling operation to ensure that no distortion is caused by the drilling pressure.

#### **CUTTING PLATE MATERIAL**

DP4® plate material may be cut to size by any one of the following methods:

- Using side and face cutter, or slitting saw, with the strip held flat and securely on a horizontal milling machine
- Cropping
- Guillotine (For widths less than 90 mm only)
- Water-jet cutting
- Laser cutting (see Health Warning)

Care must be taken to protect the bearing surface from damage and to ensure that no deformation of the strip occurs.

### 7.2 ELECTROPLATING

#### **DP4® COMPONENTS**

In order to provide some protection in mildly corrosive environments the steel back and end faces of standard range DP4® bearings are tin flashed.

DP4° can be electroplated with most of the conventional electroplating metals including the following:

- zinc ISO 2081
- nickel ISO 1456
- hard chromium ISO 1456

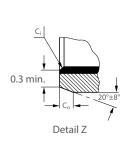
For the harder materials if the specified plating thickness exceeds approximately 5  $\mu$ m then the housing diameter should be increased by twice the plating thickness in order to maintain the correct assembled bearing bore size.

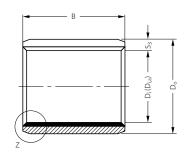
Where electrolytic attack is possible tests should be conducted to ensure that all the materials in the bearing environment are mutually compatible.

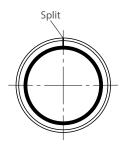
# **8 Standard Products**

# 8.1 DP4® CYLINDRICAL BUSHES - metric sizes









Dimensions and Tolerances according to ISO 3547 and GGB Specifications

### OUTSIDE Co AND INSIDE Ci CHAMFERS

WALL THICKNESS S <sub>3</sub>	C₀ MACHINED	C <sub>i</sub> (b)		
0.75	$0.5 \pm 0.3$	$0.5 \pm 0.3$	-0.1 to -0.4	
1.5	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.7	
1	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.5	

WALL THICKNESS S <sub>3</sub>	C₀ MACHINED	C <sub>i</sub> (b)	
2	$1.2 \pm 0.4$	$1.0 \pm 0.4$	-0.1 to -0.7
2.5	$1.8 \pm 0.6$	$1.2 \pm 0.4$	-0.2 to -1.0

- (a) = chamfer  $C_o$  machined or rolled at the opinion of the manufacturer
- (b) =  $C_i$  can be a radius or a chamfer in accordance with ISO 13715

PART NO.	NOMINAL DIAMETER		WALL THICKNESS  S <sub>3</sub> max.	WIDTH B max.	D	SHAFT Ø D <sub>J</sub> [H6, F7, H8] max. min.		HOUSING Ø D <sub>H</sub> [H6, H7] max.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING	CLEARANCE C <sub>D</sub> max. min.	
	Di	D <sub>o</sub>	min.					min.	max. min.		
0203DP4	2	3.5		3.25 2.75		2.000	H6	3.508	2.048		
0205DP4	2	3.3		5.25 4.75		1.994		3.500	2.000		
0303DP4				3.25 2.75		3.000 2.994			3.048 3.000	0.054 0.000	
0305DP4	3	4.5	0.750 0.730	5.25 4.75				4.508 4.500			
0306DP4				6.25 5.75	h6						
0403DP4				3.25 2.75		4.000 3.992		5.508 5.500	4.048 4.000		
0404DP4	4			4.25 3.75						0.056	
0406DP4	4	5.5		6.25 5.75						0.000	
0410DP4				10.25 9.75							
0505DP4				5.25 4.75		4.990 4.978	H7	7.015 7.000	5.055 4.990		
0508DP4	5	7		8.25 7.75							
0510DP4				10.25 9.75							
0604DP4				4.25 3.75		5.990 5.978		8.015 8.000		0.077 0.000	
0606DP4		0	1.005 0.980	6.25 5.75	f7				6.055		
0608DP4	6	8	0.500	8.25 7.75					5.990		
0610DP4				10.25 9.75							
0705DP4	_	9		5.25 4.75		6.987		9.015	7.055	0.083	
0710DP4	7		9	9		10.25 9.75		6.972		9.000	6.990

All dimensions in mm

	NOMINAL	DIAMETER	WALL THICKNESS S <sub>3</sub>	WIDTH B	D	SHAFT Ø [H6, F7, H8]		HOUSING Ø D <sub>H</sub> [H6, H7]	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN	CLEARANCE C <sub>D</sub>				
PART NO.	D <sub>i</sub>	D <sub>o</sub>	max. min.	max. min.	٠	max. min.		max. min.	H6/H7 HOUSING max. min.	max. min.				
0806DP4				6.25 5.75										
0808DP4		10		8.25 7.75		7.987		10.015	8.055	0.083				
0810DP4	8			10.25 9.75		7.972		10.000	7.990	0.003				
0812DP4				12.25 11.75										
1006DP4				12.25 11.75										
008DP4				8.25 7.75										
010DP4	10	4.2		10.25 9.75		9.987		12.018	10.058	0.086				
012DP4	10	12		12.25 11.75		9.972		12.000	9.990	0.003				
015DP4				15.25 14.75										
020DP4				20.25 19.75										
208DP4				8.25 7.75										
210DP4		14	14	14		10.25 9.75		11.984 11.966		14.018 14.000	12.058 11.990			
212DP4						12.25 11.75								
215DP4	12					15.25 14.75								
220DP4						20.25 19.75								
225DP4				25.25 24.75										
310DP4				10.25 9.75		12.984		15.018	13.058					
320DP4	13	15	1.005	20.25 19.75	f7	12.966	H7	15.000	12.990					
405DP4							0.980	5.25 4.75						
410DP4					10.25 9.75									
412DP4				12.25		12.004		16.018 16.000	14.058 13.990	0.092 0.006				
415DP4	14	16		11.75 15.25		13.984 13.966								
420DP4				14.75 20.25										
425DP4				19.75 25.25										
510DP4				24.75 10.25										
512DP4				9.75 12.25										
515DP4	15	17		11.75 15.25		14.984		17.018	15.058					
520DP4				14.75 20.25		14.966		17.000	14.990					
525DP4				19.75 25.25										
610DP4				24.75 10.25										
612DP4		16 18		9.75 12.25										
615DP4	16			11.75 15.25		15.984		18.018	16.058					
620DP4	10		16 18		14.75 20.25	.75 15.966 .25		18.000	15.990					
625DP4					19.75 25.25									
720DP4	17	19		24.75 20.25 19.75		16.984 16.966		19.021 19.000	17.061 16.990	0.095 0.006				

DARTNA	NOMINAL	DIAMETER	WALL THICKNESS S <sub>3</sub>	WIDTH B		SHAFT Ø J [H6, F7, H8]		HOUSING Ø D <sub>H</sub> [H6, H7]	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN	CLEARANCE C <sub>D</sub>	
PART NO.	D <sub>i</sub>	D <sub>o</sub>	max. min.	max. min.		max. min.		max. min.	H6/H7 HOUSING max. min.	max. min.	
1810DP4				10.25 9.75							
1815DP4	18	20	1.005	15.25 14.75		17.984		20.021	18.061	0.095	
1820DP4		20	0.980	20.25 19.75		17.966		20.000	17.990	0.006	
1825DP4				25.25 24.75							
2010DP4				10.25 9.75							
2015DP4				15.25 14.75							
2020DP4	20	23		20.25 19.75		19.980 19.959		23.021 23.000	20.071 19.990		
2025DP4				25.25 24.75							
2030DP4				30.25 29.75							
2215DP4		15.25 14.75									
2220DP4	22	25		20.25 19.75		21.980		25.021	22.071 21.990		
2225DP4	22	23		25.25 24.75		21.959		25.000		0.112	
2230DP4			1.505	30.25 29.75							
2415DP4		1.475 15.25 14.75			0.010						
2420DP4	24	27		20.25 19.75		23.980		27.021	24.071		
2425DP4	27	27		25.25 24.75		23.959		27.000	23.990		
2430DP4				30.25 29.75							
2515DP4		28			15.25 14.75	f7	f7	H7			
2520DP4				20.25 19.75		24.090		28.021 28.000	25.071 24.990		
2525DP4	25			25.25 24.75		24.980 24.959					
2530DP4				30.25 29.75							
2550DP4				50.25 49.75							
2815DP4				15.25 14.75							
2820DP4	28	32		20.25 19.75		27.980		32.025	28.085		
2825DP4	20	32		25.25 24.75		27.959		32.000	27.990		
2830DP4				30.25 29.75							
3010DP4				10.25 9.75						0.126	
3015DP4				15.25 14.75						0.010	
3020DP4	30	34	2.005 1.970	20.25 19.75		29.980		34.025	30.085		
3025DP4	30	24		25.25 24.75		29.959		34.000	29.990		
3030DP4				30.25 29.75							
3040DP4				40.25 39.75							
3220DP4				20.25 19.75							
3230DP4	32	32 36		30.25 29.75		31.975 31.950		32.085 31.990	0.135 0.015		
3240DP4				40.25 39.75							

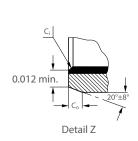
PART NO.		DIAMETER	WALL THICKNESS S <sub>3</sub> max.	WIDTH B max.	D	SHAFT Ø [H6, F7, H8] max.		HOUSING Ø D <sub>H</sub> [H6, H7] max.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING	CLEARANCE C <sub>D</sub>								
	Di	D <sub>o</sub>	min.	min.		min.		min.	max. min.	max. min.								
3520DP4				20.25 19.75						0.135 0.015								
3530DP4				30.25 29.75														
3535DP4	35	39		35.25 34.75		34.975 34.950		39.025 39.000	35.085 34.990									
3540DP4				40.25 39.75		31.550			31.550									
3550DP4			2.005	50.25 49.75														
3720DP4	37	41	1.970	20.25 19.75		36.975 36.950		41.025 41.000	37.085 36.990									
4020DP4				20.25 19.75														
4030DP4				30.25 29.75		39.975		44.025	40.085									
4040DP4	40	44		40.25 39.75		39.950		44.000	39.990									
4050DP4				50.25 49.75														
4520DP4				20.25 19.75														
4530DP4		50	50	50		30.25 29.75				50.025 50.000	45.105 44.990							
4540DP4	45					40.25 39.75		44.975 44.950				0.155 0.015						
4545DP4					45.25 44.75		77.250		30.000	44.550	0.013							
4550DP4				50.25 49.75														
5020DP4		55	55									20.25 19.75						
5030DP4						30.25 29.75	f7		H7									
5040DP4	50				40.25 39.75		49.975 49.950		55.030 55.000	50.110 49.990	0.160 0.015							
5050DP4				50.25 49.75		49.930		33.000	49.990	0.015								
5060DP4				60.25 59.75														
5520DP4				20.25 19.75														
5525DP4			2.505 2.460	25.25 24.75														
5530DP4			2.400	30.25 29.75														
5540DP4	55	60		40.25 39.75		54.970 54.940		60.030 60.000	55.110 54.990	0.170 0.020								
5550DP4				50.25 49.75		34.940		00.000	34.990	0.020								
5555DP4				55.25 54.75														
5560DP4				60.25 59.75														
6020DP4				20.25 19.75														
6030DP4				30.25 29.75														
6040DP4				40.25		50.070		65.020	60 110	0.170								
6050DP4	60	65		39.75 50.25		59.970 59.940		65.030 65.000	60.110 59.990	0.170 0.020								
6060DP4				49.75 60.25														
6070DP4				59.75 70.25 69.75														

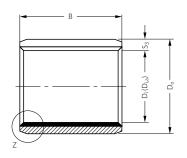
PART NO.	NOMINAL	DIAMETER	WALL THICKNESS S <sub>3</sub>	WIDTH B	D	SHAFT Ø [H6, F7, H8]		HOUSING Ø D <sub>H</sub> [H6, H7]	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING	CLEARANCE C <sub>D</sub>		
FANT NU.	Di	D <sub>o</sub>	max. min.	max. min.		max. min.		max. min.	max. min.	max. min.		
6530DP4				30.25 29.75								
6550DP4	65	70		50.25 49.75		64.970 64.940	70.030 70.000	65.110 64.990				
6570DP4				70.25 69.75		04.540		70.000	04.550			
7040DP4			2.505	40.25 39.75	_	7 69.970 69.940				0.170		
7050DP4	70	75	2.460	50.25 49.75	f7			75.030 75.000	70.110 69.990	0.020		
7070DP4				70.25 69.75		05.540		75.000	03.330			
7560DP4				60.25 59.75		74.970		80.030	75.110			
7580DP4	75	80		80.25 79.75		74.940		80.000	74.990			
8040DP4				40.50 39.50								
8060DP4				60.50 59.50		80.000		85.035	80.155			
8080DP4	80	85		80.50 79.50		79.946		85.000	80.020			
80100DP4				100.50 99.50								
8530DP4				30.50 29.50								
8560DP4	85	90		60.50 59.50	85.000 84.946	90.035 90.000	85.155 85.020					
85100DP4				100.50 99.50			90.000	65.020				
9060DP4			60.50 59.50 9	90.000		95.035	90.155					
90100DP4	90	95				89.946		95.000	90.020			
9560DP4		400	100	100	2.490	60.50 59.50		95.000	H7	100.035	95.155	0.209
95100DP4	95	100	2.440	100.50 99.50		94.946		100.000	95.020	0.020		
10050DP4				50.50 49.50				105.035 105.000				
10060DP4	100	105		60.50 59.50		100.000 99.946			100.155 100.020			
100115DP4				115.50 114.50	h8			103.000	100.020			
10560DP4				60.50 59.50		105.000		110.035	105.155			
105115DP4	105	110		115.50 114.50		104.946		110.000	105.020			
11060DP4				60.50 59.50		110.000		115.035	110.155			
110115DP4	110	115		115.50 114.50		109.946		115.000	110.020			
11550DP4				50.50 49.50		115.000		120.035	115.155			
11570DP4	115	120		70.50 69.50		114.946		120.000	115.020			
12050DP4				50.50 49.50								
12060DP4	120	125		60.50 59.50		120.000 119.946		125.040 125.000	120.210 120.070	0.264		
120100DP4			2.465	100.50 99.50		113.540		123.000	120.070	0.070		
125100DP4	125	130	2.415	100.50 99.50		125.000 124.937		130.040 130.000	125.210 125.070	0.273 0.070		
13060DP4				60.50 59.50		130.000		135.040	130.210			
130100DP4	130	135		100.50 99.50		129.937		135.000	130.210			

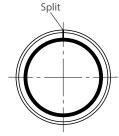
PART NO.	NOMINAL D <sub>i</sub>	DIAMETER D <sub>o</sub>	WALL THICKNESS  S <sub>3</sub> max.  min.	WIDTH B max. min.	D	SHAFT Ø <sub>J</sub> [H6, F7, H8] max. min.		HOUSING Ø D <sub>H</sub> [H6, H7] max. min.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING max. min.	CLEARANCE C <sub>D</sub> max. min.													
13560DP4	125	1.40		60.50 59.50		135.000		140.040	135.210														
13580DP4	135	140		80.50 79.50		134.937		140.000	135.070														
14060DP4	1.10	1.45		60.50 59.50		140.000		145.040	140.210														
140100DP4	140	145		100.50 99.50		139.937		145.000	140.070														
15060DP4				60.50 59.50						0.273 0.070													
15080DP4	150	150 155	155		80.50 79.50		150.000 149.937		155.040 155.000	150.210 150.070	0.07.0												
150100DP4				100.50 99.50																			
16080DP4				2.465 2.415	80.50 79.50	h8	160.000	H7	165.040	160.210													
160100DP4	160	165	2.113	100.50 99.50		159.937		165.000	160.070														
180100DP4	180	185		77.50															180.000 179.937		185.046 185.000	180.216 180.070	0.279 0.070
200100DP4	200	205				200.000 199.928		205.046 205.000	200.216 200.070	0.288													
210100DP4	210	215		100.50		210.000 209.928		215.046 215.000	210.216 210.070														
220100DP4	220	225		99.50		220.000 219.928		225.046 225.000	220.216 220.070														
250100DP4	250	255				250.000 249.928		255.052 255.000	250.222 250.070	0.294 0.070													
300100DP4	300	305				300.000 299.919		305.052 305.000	300.222 300.070	0.303 0.070													

## 8.2 DP4® CYLINDRICAL BUSHES - inch sizes









Dimensions and Tolerances according to ISO 3547 and GGB Specifications

PART NO.		TECH	NICAL D	. DATA		
		imension	s	Installation		
	Inside Ø D <sub>i</sub>	Outside Ø Do	Width B	$D_{i,a}$		
02DP402	1/8	3/16	1/8	0.1268/0.1243		
02DP403	78	716	3/16	0.1200/0.1243		
025DP4025	5/32	7/32	5/32	0.1581/0.1556		
025DP404	732	732	1/4	0.1301/0.1330		
03DP403			3/16			
03DP404	3/16	1/4	1/4	0.1893/0.1867		
03DP406			3/8			
04DP404	1/4	5/16	1/4	0.2518/0.249		
04DP406	74	716	3/8	0.2310/0.2432		
05DP406	5/16	3/8	3/8	0.3143/0.3117		
05DP408	/ 16	/8	1/2	711 C.0 (CF1 C.0		
06DP403			3/16			
06DP404			1/4			
06DP406	3/8	15/32	3/8	0.3769/0.3742		
06DP408	/8	/32	1/2	0.57 05/0.5742		
06DP410			5/8			
06DP412			3/4			
07DP408	7/16	17/32	1/2	0.4394/0.4367		
07DP412	/ 10	/32	3/4	0.7577(U.T30/		
08DP404			1/4			
08DP406			3/8			
08DP408	1/2	19/32	1/2	0.5019/0.4992		
08DP410	/2	/32	5/8	0.3013/0.4332		
08DP412			3/4			
08DP414			7/8			
09DP406			3/8			
09DP408	9/16	21/32	1/2	0.5644/0.5617		
09DP410	/ 10	/32	5/8	0.5077/0.501/		
09DP412			3/4			
10DP404			1/4			
10DP408			1/2			
10DP410	5/8	23/32	5/8	0.627/0.6242		
10DP412	, 0	,32	3/4	0.02.70.0272		
10DP414			7/8			
10DP416		25.4	1			
11DP414	11/16	25/32	7/8	0.6895/0.6867		
12DP404			1/4			
12DP406			3/8			
12DP408	3/4	7/8	1/2	0.7525/0.7493		
12DP410	, -	, 0	5/8			
12DP412			3/4			
12DP416			1			
14DP404			1/4			
14DP406			3/8			
14DP412	7/8	1	3/4	0.8775/0.874		
14DP414	, 0	'	7/8	0.05, 0.01 43		
14DP416			1			

PAKI NU.		IECH	NICAL L	AL DATA		
	[	Dimension	ıs	Installation		
	Inside	Outside	Width B	$D_{i.a}$		
4 ( DD 40 (	Ø D <sub>i</sub>	Ø D <sub>o</sub>		UI,d		
16DP406			3/8			
16DP408			1/2			
16DP412	1	1 1/8	3/4	1.0026/0.9992		
16DP416			1			
16DP420			1 1/4			
16DP424			1 ½			
18DP406			3/8			
18DP410	1 1/8	1 %2	5/8	1.1278/1.124		
18DP412			3/4			
18DP416			1			
20DP406			3/8			
20DP412			3/4			
20DP414	1 1/4	1 13/32	7/8	1.2528/1.249		
20DP416	. , .	. ,52	1	1123207 112 13		
20DP420			1 1/4			
20DP428			1 3/4			
22DP412			3/4			
22DP416	1 3/8	1 17/32	1	1.3778/1.374		
22DP424	1 /0	. /32	1 ½	1.5770/1.574		
22DP428			1 3/4			
24DP408			1/2			
24DP416			1			
24DP418	1 ½	1 21/32	1 1/8	1.5028/1.499		
24DP420	1 72	1 -732	1 1/4	1.3020/ 1.433		
24DP424			1 ½			
24DP432			2			
26DP416	1 5/8	1 25/32	1	1.6278/1.624		
26DP424	1 78	1 -732	1 ½	1.02/0/1.024		
28DP416			1			
28DP424	1 3/4	1 15/16	1 ½	1.7535/1.7489		
28DP428	1 74	1 1716	1 3/4	1./ 333/ 1./409		
28DP432			2			
30DP412			3/4			
30DP416	1 %	2 1/16	1	1.8787/1.8739		
30DP430	I 78	Z 716	1 1/8	1.0/0//1.0/39		
30DP436			2 1/4			
32DP408			1/2			
32DP416			1			
32DP424	2	23/	1 ½	2 0027/1 0000		
32DP428	2	2 3/16	1 3/4	2.0037/1.9989		
32DP432			2			
32DP440			2 1/2			
34DP408			1/2			
34DP412			3/4			
34DP416			1			
34DP424	2.1/	251	1 ½	2 1226/2 1272		
34DP428	2 1/8	2 5/16	1 3/4	2.1326/2.1262		
34DP432			2			
34DP448			3			
2400466			4			

PART NO.		TECH	NICAL D	ATA
	_	imension	S	Installation
	Inside Ø Di	Outside Ø Do	Width B	$D_{i,a}$
40DP416			1	
40DP426			1 1/8	
40DP432			2	
40DP440			2 ½	
40DP448	2 ½	2 11/16	3	2.5077/2.5013
40DP456	_ ,_	_ ,	3 ½	
40DP460			3 3/4	
40DP464 40DP472			4 1/2	
40DP472 40DP476			4 1/2	
44DP432			2	
44DP436			2 1/4	
44DP440			2 1/2	
44DP448			3	
44DP456	2.2/	2.15/	3 ½	2 75///2 7502
44DP460	2 3/4	2 15/16	3 3/4	2.7566/2.7502
44DP464			4	
44DP472			4 1/2	
44DP476			4 3/4	
44DP480			5	
48DP432			2	
48DP436			2 1/4	
48DP440		3 3/16	2 ½	
48DP448			3	
48DP456	3		3 ½	3.0068/3.0002
48DP460			3 3/4	
48DP464 48DP472			4 1/2	
48DP476			4 1/2	
48DP480			5	
52DP432			2	
52DP438			2 1/4	
52DP440			2 ½	
52DP448			3	
52DP456	3 1/4	3 7/16	3 ½	3.2568/3.2502
52DP460	J 74	J 1/16	3 ¾	5.2300/5.2302
52DP464			4	
52DP472			4 1/2	
52DP476			4 3/4	
52DP480			5	
56DP432			2 3/	
56DP438 56DP440			2 3/8	
56DP440 56DP448			3	
56DP446			3 ½	
56DP460	3 ½	3 11/16	3 3/4	3.5068/3.5002
56DP464			4	
56DP472			4 1/2	
56DP476			4 3/4	
56DP480			5	

Inside	PART NO.			NICAL D		PART NO.			NICAL D	
SBPP436		_		IS	Installation				ıs	Inst
58BP2440         38BP2440         2 ½         2 ½         2 ½         2 ½         2 ½         2 ½         2 ½         2 ½         3 3         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         3 3 ½         4 ½				Width B	$D_{i,a}$				Width B	
SSBP440   SSBP456   SSBP460   SSBP460   SSBP460   SSBP466   SSBP466   SSBP466   SSBP466   A 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	58DP432			2		76DP432			2	
S8BP446   S8BP466   S8BP466   S8BP466   S8BP466   S8BP467   S8BP476   S8BP486   S8BP										
S8BP466   3 %   3 1%   3 1%   3 4										
58BP460 SBP464 SBP476 SBP476 SBP480										
S8DP464   S8DP476		3 5/8	3 13/16		3.6318/3.6252		4 3/4	4 15/16		4.75
58DP476         58DP476         4 ½         76DP472         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         4 ½         60DP432         60DP432         60DP436         5         80DP436         80DP448         60DP460         60DP460         60DP460         80DP448         80DP440         2 ½         60DP460         60DP460         5         3 ½         3 ½         3 ½         4 ½         80DP466         5         3 ½         2 ½         4 ½         4 ½         2 ½         4 ½										
58DP480         4 34         5         76DP476         4 34         4 34         5         60DP436         60DP436         60DP440         2 2 14         80DP440         80DP440         2 2 14         80DP440         2 2 14         80DP440         80DP440         2 2 14         80DP440         3 3         80DP440         3 3         80DP440         3 34         80DP440         3 34         80DP440         4 4 14         80DP440         4 4 14         80DP440         4 4 14         4 14         80DP440         4 4 14         4 1										
S8DP480										
CODP432   CODP448   CODP449   CODP										
SODP440	60DP432					80DP432				
SODP448	60DP436			2 1/4		80DP436			2 1/4	
60DP460 60DP460 60DP460 60DP467 60DP472 60DP472 60DP473 60DP474 60DP480 5 5 60DP480 5 60DP480 5 60DP480 60DP480 60DP480 60DP480 60DP480 60DP480 60DP480 60DP480 64DP432 64DP432 64DP436 64DP440 64DP440 64DP446 64DP464 64DP476 64DP480 64DP48	60DP440			2 1/2		80DP440			2 1/2	
60DP460 60DP464 60DP472 60DP476 60DP480 60DP480 64DP480 64DP48	60DP448			3		80DP448			3	
600P464         4         800P464         4         4         4         800P466         4		3 3/4	3 15/16		3 7568/3 7502		5	5 3/16		5.004
60DP472 60DP476 60DP480 5 60DP480 64DP436 64DP436 64DP440 64DP440 64DP440 64DP460 64DP460 64DP472 64DP472 64DP476 64DP480 5 68DP480 68DP48		3 /4	3 /10		3.7300/3.7302		5	3 /10		3.003
60DP476 60DP480 60DP480 60DP480 60DP480 60DP480 60DP480 60DP480 64DP432 2 2 44 64DP432 64DP448 64DP448 64DP456 64DP460 64DP460 64DP460 64DP460 64DP472 64DP476 64DP480 5 5 68DP480 68D				-					-	
60DP480 64DP432 64DP436 64DP440 64DP448 64DP456 64DP460 64DP460 64DP460 64DP472 64DP476 64DP470 64DP480 68DP480 68DP48										
64DP432 64DP440 64DP440 64DP440 64DP440 64DP440 64DP440 64DP460 64DP460 64DP460 64DP460 64DP460 64DP460 64DP470 4 3/4										
64DP436 64DP448 64DP448 64DP456 64DP460 64DP460 64DP460 64DP477 64DP476 64DP476 64DP478 64DP480 64DP48										
64DP440 64DP460 64DP460 64DP460 64DP460 64DP460 64DP460 64DP460 64DP460 64DP472 64DP476 64DP476 64DP480 68DP430 68DP430 68DP440 68DP440 68DP440 68DP440 68DP440 68DP440 68DP460 68DP46										
64DP448 64DP456 64DP460 64DP460 64DP472 64DP476 64DP480 64DP480 65 68DP436 68DP440 68DP440 68DP440 68DP440 68DP440 68DP440 68DP440 68DP464 68DP476 68DP464 68DP464 68DP464 68DP464 68DP476 68DP464 68DP464 68DP476 68DP464 68DP480 5 5 8BDP460 8BDP480 5 5 70DP436 70DP480 70DP480 70DP460 70DP460 70DP460 70DP460 70DP464 70DP472 70DP460 70D	64DP440									
64DP460 64DP464 64DP472 64DP476 64DP476 64DP476 64DP480 5 5 64DP480 68DP432 68DP432 68DP448 68DP448 68DP448 68DP448 68DP456 68DP448 68DP464 64DP472 68DP464 68DP472 68DP464 68DP464 68DP472 68DP464 68DP464 68DP472 68DP464 68DP464 68DP472 68DP464 68DP464 68DP464 68DP464 68DP472 68DP466 68DP464 68DP472 68DP466 68DP464 68DP464 68DP472 68DP466 68DP464 68DP472 68DP466 68DP464 68DP472 68DP466 68DP466 68DP464 68DP472 68DP466 68DP466 68DP466 68DP466 68DP466 68DP466 68DP472 68DP466 68DP472 68DP472 68DP476 68DP472 68DP476 68DP472 68DP476 68	64DP448					84DP448				
64DP464 64DP472 64DP476 64DP480 65 68DP432 68DP436 68DP432 68DP436 68DP440 68DP480 68D	64DP456	1	4 34.	3 ½	4 0069/4 0002	84DP456	E 1/.	E 74.	3 ½	5 25
64DP476 64DP476 64DP480 65BP432 68DP436 68DP436 68DP440 68DP480 68DP448 68DP440 68DP480 70DP432 70DP432 70DP436 70DP440 70DP440 70DP440 70DP472 70DP476 70DP480 72DP440 72DP476 72DP440 72DP476 72DP460 72DP480 72DP48	64DP460	4	4 716	3 ¾	4.0000/4.0002	84DP460	J 74	3 716	3 ¾	3.23
64DP476 64DP480 68DP432 68DP432 68DP440 68DP440 68DP440 68DP440 68DP456 68DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP464 70DP472 70DP460 70DP472 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP470 70DP460 70DP460 70DP460 70DP470 70DP460 70DP460 70DP460 70DP460 70DP470 70DP460 70DP460 70DP460 70DP470 70DP460 70DP470 70DP460 70DP46										
64DP480 68DP432 68DP440 68DP464 68DP472 68DP460 68DP480 70DP480 70DP480 70DP460 70DP472 70DP472 70DP476 70DP460 70DP460 70DP460 70DP460 70DP472 70DP470 70DP460 70DP460 70DP472 70DP470 70DP460 70DP470 70DP460 70DP470 70DP460 70DP470 70DP460 70DP470 70DP460 70DP470 70DP47										
68DP432 68DP440 68DP440 68DP440 68DP440 68DP456 68DP460 68DP464 68DP472 68DP472 68DP470 68DP480 70DP480 70DP440 70DP440 70DP460 70DP470 70DP460 70DP460 70DP460 70DP470 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP470 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP460 70DP470 70DP460 70DP460 70DP470 70DP460 70DP460 70DP470 70DP460 70DP470 70DP460 70DP470 70DP460 70DP470 70DP460 70DP470 70DP47									. , .	
68DP440 68DP448 68DP446 68DP446 68DP460 68DP460 68DP464 68DP472 68DP476 68DP480 70DP432 70DP436 70DP440 70DP440 70DP440 70DP476 70DP464 70DP472 70DP476 70DP480 72DP486 72DP486 72DP466 72DP466 72DP466 72DP466 72DP466 72DP476										
SBDP440   SDPP432   SBDP440   SDPP432   SDPP432   SDPP432   SDPP432   SDPP440   SDPP										
Sedding				_ , ,					_ , .	
68DP456 68DP460 68DP464 68DP472 68DP476 68DP480 70DP432 70DP436 70DP440 70DP446 70DP472 70DP476 70DP476 70DP476 70DP476 70DP472 7DP476 7DP480 7DP480 7DP480 7DP480 7DP480 7DP480 7DP480 7DP480 7DP4980										
68DP464 68DP472 68DP476 68DP476 68DP480 5 70DP432 70DP436 70DP448 70DP448 70DP464 70DP472 70DP476 70DP472 70DP476 4 3/4 4 3/4 70DP476 70DP472 70DP480	68DP456	4.1/	4.7/	3 1/2	4 3560/4 3503	88DP456	F 1/	F 11/	3 ½	F F0
68DP472 68DP476 68DP480 5 70DP432 70DP432 70DP448 70DP440 70DP460 70DP464 70DP472 70DP476 70DP480 5 70DP480 70	68DP460	4 1/4	4 1/16	3 3/4	4.2508/4.2502	88DP460	<b>5</b> ½	J 1/16	3 3/4	5.50
68DP476 68DP480 5 70DP432 70DP436 70DP440 70DP448 70DP456 70DP464 70DP472 70DP476 70DP480 5 2 2 ½ 3 ¾ 4 %6 3 ½ 3 ¾ 4 %6 4 ½ 4 ½ 70DP460 70DP472 70DP480 5 72DP480 72DP480 72DP480 72DP480 72DP460 72DP460 72DP460 72DP460 72DP460 72DP460 72DP476 72DP460 72DP476	68DP464			4		88DP464			4	
S8DP480				- , -					. , .	
70DP432 70DP440 70DP440 70DP456 70DP460 70DP464 72DP472 72DP476 4 ½ 4 ½ 4 ½ 4 ½ 4 ½ 4 ½ 4 ½ 4 ½ 4 ½ 4										
70DP436 70DP440 70DP448 70DP456 70DP460 70DP460 70DP460 70DP460 70DP460 70DP472 70DP476 70DP476 70DP476 70DP480 70DP48										
70DP440 70DP456 70DP460 70DP460 70DP460 70DP476 70DP476 70DP476 70DP476 70DP476 70DP480 72DP432 72DP436 72DP440 72DP440 72DP460 72DP476 72DP476 72DP476										
70DP448 70DP456 70DP460 70DP460 70DP464 70DP464 70DP472 70DP476 70DP480 72DP480 72DP48									_ , .	
70DP456 70DP460 70DP464 70DP464 70DP472 70DP476 70DP480 5 72DP432 72DP436 72DP480 72DP460 72DP460 72DP460 72DP460 72DP470 72DP470 72DP470 72DP470 72DP470 72DP470 72DP470 72DP470 72DP470										
70DP460 70DP464 70DP472 70DP476 4 ½ 92DP476 4 ½ 92DP476 4 ¼ 92DP476 4 ¾ 70DP480 5 72DP480 5 72DP432 72DP436 72DP440 72DP440 72DP440 72DP460 4 ½ 4 ¼ 4 ½ 96DP440 96DP464 4 ½ 96DP464 72DP472 72DP464 4 ½ 96DP464 72DP472 72DP476	70DP456	4.2/	4.07		4 2010 /4 2752	92DP456	F 3/	F 15/		r 7r
70DP472 70DP476 70DP476 70DP480 5 92DP480 5 72DP430 72DP432 72DP436 72DP440 72DP440 72DP448 72DP456 72DP464 72DP464 72DP464 72DP472 72DP476 4 ½ 4 ¾ 4 1¼6 3 ½ 4 .5068/4.5002 96DP440 96DP464 96DP464 96DP464 96DP472 96DP472 96DP476	70DP460	4 %	4 %16	3 3/4	4.3818/4.3/52	92DP460	5 ¾	5 13/16	3 3/4	5./5
70DP476 70DP480 5 92DP476 92DP480 5 72DP432 72DP436 72DP440 72DP440 72DP448 72DP456 72DP460 72DP464 72DP476										
70DP480						92DP472				
72DP432 72DP436 72DP440 72DP448 72DP456 72DP460 72DP460 72DP464 72DP472 72DP476										
72DP436 72DP440 72DP448 72DP456 72DP460 72DP460 72DP464 72DP472 72DP476  4 ½ 4 ⅓ 4 ⅓										
72DP440 72DP456 72DP456 72DP460 72DP464 72DP464 72DP472 72DP476  4 ½ 4 ½ 4 ½ 4 ½ 4 ¾ 4 ½ 96DP456 96DP464 96DP464 4 ½ 96DP472 96DP472 96DP476										
72DP448 72DP456 72DP460 72DP464 72DP472 72DP476										
72DP456     4 ½     4 1½     3 ½     4.5068/4.5002     96DP456     6 ¾6     3 ½     3 ½       72DP464     4     4     96DP464     4     4       72DP472     4 ¾     96DP472     4 ½     4 ½       72DP476     4 ¾     96DP476     4 ¾										
72DP460 4 1/2 4 1/16 3 3/4 4.5008/4.5002 96DP460 6 6 7/16 3 3/4 6.000 72DP464 72DP472 4 1/2 96DP472 96DP476 6 6 7/16 3 3/4 4.5008/4.5002 96DP472 4 1/2 96DP476 96DP476 96DP476 4 3/4										
72DP464     4     96DP464     4       72DP472     4 ½     96DP472     4 ½       72DP476     4 ¾     96DP476     4 ¾		4 1/2	4 11/16		4.5068/4.5002		6	6 3/16		6.00
72DP472     4 ½     96DP472     4 ½       72DP476     4 ¾     96DP476     4 ¾										
<b>72DP480</b> 5 <b>96DP480</b> 5										
	72DP480			5		96DP480			5	

PART NO.		TECH	NICAL D	ATA
	[	imension		Installation
	Inside Ø Di	Outside Ø Do	Width B	D <sub>i,a</sub>
100DP432	~ -1	~ - 0	2	
100DP436			2 1/4	
100DP440			2 1/2	
100DP448			3	
100DP456	6 1/4	6 7/16	3 ½	6.257/6.2502
100DP460	0 74	0 716	3 ¾	0.237/0.2302
100DP464			4	
100DP472			4 1/2	
100DP476			4 3/4	
100DP480			5	
104DP432			2	
104DP436			2 1/4	
104DP440			2 ½	
104DP448			3	
104DP456	6 1/2	6 11/16	3 ½	6.507/6.5002
104DP460	0 /2		3 ¾	0.50770.5002
104DP464			4	
104DP472			4 1/2	
104DP476			4 3/4	
104DP480			5	
108DP432			2	
108DP436			2 1/4	
108DP440			2 ½	
108DP448			3	
108DP456	6 3/4	6 15/16	3 ½	6.757/6.7502
108DP460			3 ¾	
108DP464			4	
108DP472			4 ½	
108DP476			4 3/4	
108DP480			5	
112DP432			2	
112DP436			2 1/4	
112DP440			2 ½	
112DP448 112DP456				
112DP456 112DP460	7	7 3/16	3 ½ 3 3/4	7.0026/6.9956
112DP460 112DP464			3 %	
112DP464 112DP472			4 1/2	
112DP472			4 1/2	
112DP476			5	
11207480			Э	

Installation

 $D_{i,a}$ 

4.7572/4.7502

5.0056/4.9988

5.257/5.2502

5.507/5.5002

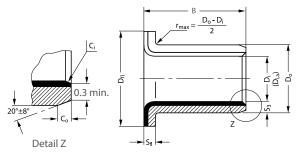
5.757/5.7502

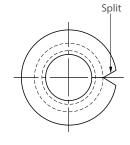
6.007/6.0002

All dimensions in inch

## 8.3 DP4® FLANGED BUSHES - metric sizes







Dimensions and Tolerances according to ISO 3547 and GGB Specifications

#### OUTSIDE Co AND INSIDE Ci CHAMFERS

WALL THICKNESS S <sub>3</sub>	C₀ MACHINED		C <sub>i</sub> (b)
0.75	$0.5 \pm 0.3$	$0.5 \pm 0.3$	-0.1 to -0.4
1.5	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.7
1	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.5

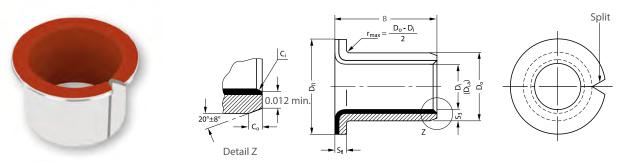
WALL THICKNESS S <sub>3</sub>	C₀ MACHINED		C <sub>i</sub> (b)
2	$1.2 \pm 0.4$	$1.0 \pm 0.4$	-0.1 to -0.7
2.5	$1.8 \pm 0.6$	$1.2 \pm 0.4$	-0.2 to -1.0

- (a) = chamfer  $C_o$  machined or rolled at the opinion of the manufacturer
- (b) =  $C_i$  can be a radius or a chamfer in accordance with ISO 13715

PART NO.		IINAL METER D <sub>o</sub>	WALL THICK- NESS S <sub>3</sub> max. min.	FLANGE THICKN. S <sub>fl</sub> max. min.	FLANGE Ø D <sub>fl</sub> max. min.	WIDTH B max. min.		SHAFT Ø H6, F7, H8] max. min.		USING Ø [H6, H7] max. min.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING max. min.	CLEARANCE C <sub>D</sub> max. min.										
BB0304DP4	3	4.5	0.750	0.80	7.50 6.50	4.25 3.75	h.c	3.000 2.994	H6	4.508 4.500	3.048 3.000	0.054 0.000										
BB0404DP4	4	5.5	0.730	0.70	9.50 8.50	4.25 3.75	h6	4.000 3.992	110	5.508 4.500	4.048 4.000	0.056 0.000										
BB0505DP4	5	7			10.50 9.50	5.25 4.75		4.990 4.978		7.015 7.000	5.055 4.990	0.077 0.000										
BB0604DP4					12.50	4.25 3.75		5.990		8.015	6.055	0.077										
BB0608DP4	6	8			11.50	8.25 7.75		5.978		8.000	5.990	0.000										
BB0806DP4						5.75 5.25																
BB0808DP4	8	10			15.50 14.50	7.75 7.25		7.987 7.972		10.015 10.000	8.055 7.990	0.083 0.003										
BB0810DP4					11.50	9.75 9.25		7.372		10.000	7.550	0.003										
BB1007DP4																7.25 6.75						
BB1009DP4			1.005	1.05	18.50	9.25 8.75	67	9.987		H7 12.018	10.058	0.086										
BB1012DP4	10	10	10	10	0 12	0.980	0.80	17.50	12.25 11.75	f7	9.972 H	H/	12.000	9.990	0.003							
BB1017DP4						17.25 16.75																
BB1207DP4						7.25 6.75																
BB1209DP4	12	1.4			20.50	9.25 8.75		11.984		14.018	12.058											
BB1212DP4	12	! 14			19.50	12.25 11.75		11.966		14.000	11.990	0.092										
BB1217DP4						17.25 16.75						0.006										
BB1412DP4		16			22.50	12.25 11.75		13.984		16.018	14.058											
BB1417DP4	14	16			21.50	17.25 16.75		13.966		16.000	13.990											

PART NO.		IINAL METER D <sub>o</sub>	WALL THICK- NESS S <sub>3</sub> max. min.	FLANGE THICKN. S <sub>fl</sub> max. min.	FLANGE Ø D <sub>fl</sub> max. min.	WIDTH B max. min.		SHAFT Ø [H6, F7, H8] max. min.	HC D <sub>H</sub>	OUSING Ø   [H6, H7]   max.   min.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING max. min.	CLEARANCE C <sub>D</sub> max. min.	
BB1509DP4						9.25 8.75							
BB1512DP4	15	17			23.50 22.50	12.25 11.75		14.984 14.966		17.018 17.000	15.058 14.990		
BB1517DP4					22.30	17.25 16.75		14.500		17.000	14.550	0.092 0.006	
BB1612DP4			1.005	1.05	24.50	12.25 11.75		15.984		18.018	16.058	0.000	
BB1617DP4	16	18	0.980	0.80	23.50	17.25 16.75		15.966		18.000	15.990		
BB1812DP4						12.25 11.75							
BB1817DP4	18	20			26.50 25.50	17.25 16.75		17.984 17.966		20.021 20.000	18.061 17.990	0.095 0.006	
BB1822DP4					23.30	22.25 21.75		171500		20.000		0.000	
BB2012DP4						11.75 11.25							
BB2017DP4	20	23			30.50 29.50	16.75 16.25		19.980 19.959		23.021 23.000	20.071 19.990		
BB2022DP4			1.505				21.75 21.25	f7		H7			0.112
BB2512DP4			1.475		35.50 34.50	11.75 11.25	17		П			0.010	
BB2517DP4	25	28				16.75 16.25		24.980 24.959		28.021 28.000	25.071 24.990		
BB2522DP4						21.75 21.25							
BB3016DP4	30	34			42.50	16.25 15.75		29.980		34.025	30.085	0.126	
BB3026DP4	30	34			41.50	26.25 25.75		29.959		34.000	29.990	0.010	
BB3516DP4	35	39	2.005	2.10	47.50	16.25 15.75		34.975		39.025	35.085		
BB3526DP4	33	39	1.970	1.80	46.50	26.25 25.75		34.950		39.000	34.990	0.135 0.015	
BB4016DP4	40	4.4			53.50	16.25 15.75		39.975		44.025	40.085		
BB4026DP4	40	44			52.50	26.25 25.75		39.950		44.000	39.990		
BB4516DP4	45	F.O.	2.505	2.60	58.50	16.25 15.75		44.975		50.025	45.105	0.155	
BB4526DP4	45	50	2.460	2.30	57.50	26.25 25.75		44.950		50.000	44.990	0.015	

## 8.4 DP4® FLANGED BUSHES - inch sizes



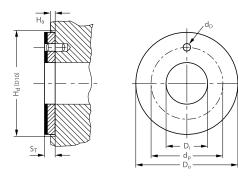
Dimensions and Tolerances according to ISO 3547 and GGB Specifications

PART NO.			TEC	HNICA	L DATA	
			Dimens			Installation
	Inside Ø D <sub>i</sub>	Outside Ø D <sub>o</sub>	Width B	Flange Ø D <sub>fl</sub>	Flange thickness	$D_{i,a}$
06FDP404			1/4			
06FDP406	3/8	1/2	3/8	2/3		0.3779/0.3752
06FDP408	-78	72	1/2	73		0.3779/0.3732
06FDP412			3/4			
08FDP404			1/4			
08FDP406	1/2	3/5	3/8	4/5	0.047-0.039	0.5029/0.5002
08FDP408	/2	/3	1/2	/3	0.047 0.037	0.3027/0.3002
08FDP412			3/4			
10FDP406			3/8			
10FDP408	5/8	5/7	1/2	1		0.628/0.6252
10FDP410	70	//	5/8			0.020/ 0.0232
10FDP412			3/4			
12FDP406			3/8			
12FDP408	3/4	7/8	1/2	1 1/4		0.7534/0.7502
12FDP412	, .	,,,	3/4			01733 17017302
12FDP416			1			
14FDP408			1/2			
14FDP412	7/8	1	3/4	1 1/4	0.063-0.055	0.8784/0.8752
14FDP416	, -		1			
14FDP420			1 1/4			
16FDP408			1/2			
16FDP412	1	1 1/8	3/4	1 3/8		1.0034/1.0002
16FDP416			1			
16FDP420			1 1/4			
20FDP416	4.1/	4.2/	1	4.3/		4 354/4 3533
20FDP420	1 1/4	1	1 1/4	1 3/4		1.254/1.2502
20FDP424			1 ½		0.078-0.07	
24FDP416	1.1/	1.2/	1	2		1 504/1 5000
24FDP424	1 ½	1 3/3	1/2	2		1.504/1.5002
24FDP432			2			

All dimensions in inch

## 8.5 DP4® THRUST WASHERS - metric sizes



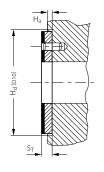


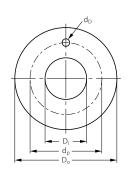
Dimensions and Tolerances according to ISO 3547 and GGB Specifications

DARTNO	INSIDE D			DIAMETER D <sub>o</sub>	THICKNESS S <sub>T</sub>	DOWI Ø d <sub>D</sub>	EL HOLE PCD Ø d <sub>d</sub>	RECESS DEPTH H <sub>a</sub>
PART NO.	max.	min.	max.	min.	max. min.	max. min.	max. min.	max. min.
WC08DP4	10.25	10.00	20.00	19.75		No Hole	No Hole	
WC10DP4	12.25	12.00	24.00	23.75		1.875 1.625	18.12 17.88	
WC12DP4	14.25	14.00	26.00	25.75			20.12 19.88	
WC14DP4	16.25	16.00	30.00	29.75		2.375 2.125	22.12 21.88	
WC16DP4	18.25	18.00	32.00	31.75		2.123	25.12 24.88	
WC18DP4	20.25	20.00	36.00	35.75		3.375 3.125	28.12 27.88	1.20 0.95
WC20DP4	22.25	22.00	38.00	37.75	1.50 1.45		30.12 29.88	
WC22DP4	24.25	24.00	42.00	41.75			33.12 32.88	
WC24DP4	26.25	26.00	44.00	43.75			35.12 34.88	
WC25DP4	28.25	28.00	48.00	47.75			38.12 37.88	
WC30DP4	32.25	32.00	54.00	53.75			43.12 42.88	
WC35DP4	38.25	38.00	62.00	61.75			50.12 49.88	
WC40DP4	42.25	42.00	66.00	65.75		4.375 4.125	54.12 53.88	
WC45DP4	48.25	48.00	74.00	73.75		25	61.12 60.88	
WC50DP4	52.25	52.00	78.00	77.75	2.00 1.95		65.12 64.88	1.70 1.45
WC60DP4	62.25	62.00	90.00	89.75			76.12 75.88	

## 8.6 DP4® THRUST WASHERS - inch sizes





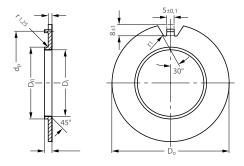


Dimensions and Tolerances according to ISO 3547 and GGB Specifications

PART NO.				TECHNIC	AL DATA						
	Non	ninal Dimens	ions	Installation							
	Inside Ø D <sub>i</sub>	Outside Ø D <sub>o</sub>	Thickness S <sub>T</sub>	Dowel hole	Pitch circle Ø	rd max	rd min	Recess depth			
G06DP4	0.5	0.875		0.072	0.687						
G07DP4	0.562	1		0.072	0.781						
G08DP4	0.625	1.125			0.875						
G09DP4	0.687	1.187		0.104	0.937			0.05-0.04			
G10DP4	0.75	1.25		0.104	1	0.05	0.04				
G11DP4	0.812	1.375			1.094						
G12DP4	0.875	1.5	0.0605	0.135	1.187						
G14DP4	1	1.75	0.0003		1.375						
G16DP4	1.125	2			1.562						
G18DP4	1.25	2.125		0.166	1.687						
G20DP4	1.375	2.25			1.812						
G22DP4	1.5	2.5			2						
G24DP4	1.625	2.625			2.125						
G26DP4	1.75	2.75		0.197	2.25						
G28DP4	2	3		0.197	2.5			0.08-0.07			
G30DP4	2.125	3.125	0.0915		2.625	0.08	0.07				
G32DP4	2.25	3.25			2.75						

## 8.7 DP4® FLANGED WASHERS - metric sizes



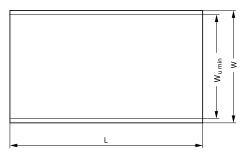


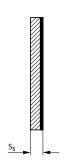
Dimensions and Tolerances according to ISO 3547 and GGB Specifications

PART NO.	INSIDE DIAMETER	OUTSIDE DIAMETER	FLANGE Ø	LOCATION Ø
	D <sub>i</sub>	D <sub>o</sub>	D <sub>fl</sub>	d <sub>P</sub>
	max.	max.	max.	max.
	min.	min.	min.	min.
BS40DP4	40.7	75.0	44.00	65.0
	40.2	74.5	43.90	64.5
BS50DP4	51.5	85.0	55.00	75.0
	51.0	84.5	54.88	74.5
BS60DP4	61.5	95.0	65.00	85.0
	61.0	94.5	64.88	84.5
BS70DP4	71.5	110.0	75.00	100.0
	71.0	109.5	74.88	99.5
BS80DP4	81.5	120.0	85.00	110.0
	81.0	119.5	84.86	109.5
BS90DP4	91.5	130.0	95.00	120.0
	91.0	129.5	94.86	119.5
BS100DP4	101.5	140.0	105.00	130.0
	101.0	139.5	104.86	129.5

## 8.8 DP4® PLATES - metric sizes



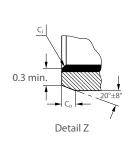


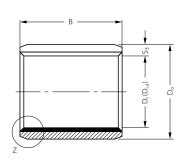


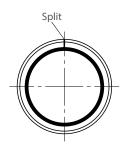
PART NO.	LENGTH L max. min.	TOTAL WIDTH W	USABLE WIDTH WU min	THICKNESS S <sub>S</sub> max. min.
S10190DP4				1.01 0.97
S15190DP4		200	190	1.52 1.48
S20190DP4	503 500			1.98 1.94
S25240DP4	300	254	240	1.52 1.48
S15240DP4		254	240	2.46 2.42

## 8.9 DP4-B CYLINDRICAL BUSHES - metric sizes









Dimensions and Tolerances according to ISO 3547 and GGB Specifications  $\,$ 

## OUTSIDE Co AND INSIDE Ci CHAMFERS

WALL THICKNESS S <sub>3</sub>	C₀ MACHINED		C <sub>i</sub> (b)
0.75	$0.5 \pm 0.3$	$0.5 \pm 0.3$	-0.1 to -0.4
1.5	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.7
1	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.5

WALL THICKNESS S <sub>3</sub>	C₀ MACHINED	C <sub>i</sub> (b)	
2	$1.2 \pm 0.4$	$1.0 \pm 0.4$	-0.1 to -0.7
2.5	$1.8 \pm 0.6$	$1.2 \pm 0.4$	-0.2 to -1.0

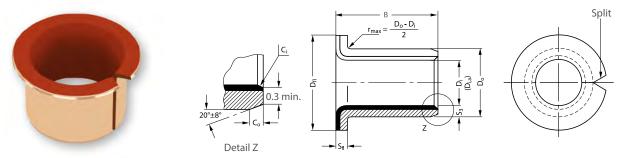
- (a) = chamfer  $C_0$  machined or rolled at the opinion of the manufacturer
- (b) =  $C_i$  can be a radius or a chamfer in accordance with ISO 13715

PART NO.	NOMINAL	DIAMETER	WALL THICKNESS S <sub>3</sub>	WIDTH B	D	SHAFT Ø <sub>J</sub> [H6, F7, H8]		HOUSING Ø D <sub>H</sub> [H6, H7]	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING	CLEARANCE C <sub>D</sub>	
TART NO.	Di	D <sub>o</sub>	max. min.	max. min.		max. min.		max. min.	max. min.	max. min.	
0203DP4B	2	2.5		3.25 2.75		2.000		3.508	2.048		
0205DP4B	2	3.5		5.25 4.75		1.994		3.500	2.000	0.054 0.000	
0306DP4B	3	4.5	0.750 0.730	6.25 5.75 h6	3.000 2.994	H6	4.508 4.500	3.048 3.000	0.000		
0404DP4B			0.730	4.25 3.75		4.000		5.508	4.048	0.056	
0406DP4B	4	5.5		6.25 5.75		3.992		5.500	4.000	0.000	
0505DP4B		_		5.25 4.75		4.990		7.015	5.055		
0510DP4B	5	7		10.25 9.75		4.978		7.000	4.990		
0606DP4B				6.25 5.75						0.077 0.000	
0608DP4B	6	8	8		8.25 7.75		5.990 5.978		8.015 8.000	6.055 5.990	0.000
0610DP4B				10.25 9.75		3.57 0		0.000	3.550		
0808DP4B				8.25 7.75							
0810DP4B	8	10	1.005	10.25 9.75	f7	7.987 7.972		10.015	8.055 7.990	0.083	
0812DP4B			0.980	12.25 11.75	1/	7.372	H7	10.000	7.550	0.003	
1010DP4B	10	12		10.25 9.75		9.987		12.018	10.058	0.086	
1015DP4B	10	12		15.25 14.75		9.972		12.000	9.990	0.003	
1208DP4B				8.25 7.75							
1210DP4B				10.25 9.75		11.984 11.966	14.018	12.058	0.092		
1212DP4B	12	12 14		12.25 11.75			14.000 11.990		0.006		
1215DP4B				15.25 14.75							

	NOMINAL	DIAMETER	WALL THICKNESS S <sub>3</sub>	WIDTH B		SHAFT Ø 1 [H6, F7, H8]		HOUSING Ø D <sub>H</sub> [H6, H7]	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN	CLEARANCE C <sub>D</sub>
PART NO.	D <sub>i</sub>	D <sub>o</sub>	max. min.	max. min.		max. min.		max. min.	H6/H7 HOUSING max. min.	max. min.
1410DP4B				10.25 9.75						
1415DP4B	14	16		15.25 14.75		13.984 13.966		16.018 16.000	14.058 13.990	
1420DP4B				20.25 19.75						
1515DP4B	15	17		15.25 14.75		14.984		17.018	15.058	0.092 0.006
1525DP4B	15	17	1.005 0.980	25.25 24.75		14.966		17.000	14.990	
1615DP4B	16	18		15.25 14.75		15.984		18.018	16.058	
1625DP4B	10	10		25.25 24.75		15.966		18.000	15.990	
1820DP4B	10	20		20.25 19.75		17.984		20.021	18.061	0.095
1825DP4B	18	20		25.25 24.75		17.966		20.000	17.990	0.006
2015DP4B				15.25 14.75						
2020DP4B	20	23		20.25 19.75		19.980		23.021	20.071	
2025DP4B	20	25		25.25 24.75		19.959		23.000	19.990	
2030DP4B				30.25 29.75						
2215DP4B			1.505 1.475	15.25 14.75					0.112 0.010	
2220DP4B	22	25		20.25 19.75		21.980 21.959		25.021 25.000	22.071 21.990	
2225DP4B				25.25 24.75						
2515DP4B	25	28		15.25 14.75		24.980	H7	28.021	25.071	
2525DP4B	25	20		25.25 24.75	f7	24.959		28.000	24.990	
2830DP4B	28	32		30.25 29.75		27.980 27.959		32.025 32.000	28.085 27.990	0.126 0.010
3020DP4B				20.25 19.75						
3030DP4B	30	34		30.25 29.75		29.980 29.959		34.025 34.000	30.085 29.990	
3040DP4B			2.005	40.25 39.75						
3520DP4B	35	39	1.970	20.25 19.75		34.975		39.025	35.085	
3530DP4B	33	39		30.25 29.75		34.950		39.000	34.990	0.135
4030DP4B	40	44		30.25 29.75		39.975		44.025	40.085	0.015
4050DP4B	40	44		50.25 49.75		39.950		44.000	39.990	
4530DP4B	45	50		30.25 29.75		44.975		50.025	45.105	0.155
4550DP4B	45	50		50.25 49.75		44.950		50.000	44.990	0.015
5040DP4B	50		2.505 2.460	40.25 39.75		49.975		55.030	50.110	0.160
5060DP4B	50	55		60.25 59.75		49.950		55.000	49.990	0.015
5540DP4B	55	60		40.25 39.75		54.970 54.940		60.030 60.000	55.110 54.990	
6040DP4B				40.25 39.75						
6050DP4B	60	65		50.25 49.75		59.970		65.030	60.110	0.170 0.020
6060DP4B	60	65		60.25 59.75		59.940		65.000	59.990	
6070DP4B				70.25 69.75						

PART NO.	NOMINAL D <sub>i</sub>	DIAMETER D <sub>o</sub>	WALL THICKNESS  S <sub>3</sub> max.  min.	WIDTH B max. min.	D	SHAFT Ø D <sub>J</sub> [H6, F7, H8] max. min.		D <sub>J</sub> [H6, F7, H8] max. min.		D <sub>J</sub> [H6, F7, H8] max. min.		D <sub>J</sub> [H6, F7, H8] max. min.		D <sub>J</sub> [H6, F7, H8] max.		D <sub>J</sub> [H6, F7, H8] max.		D <sub>J</sub> [H6, F7, H8] max.		D <sub>J</sub> [H6, F7, H8] max. min.		HOUSING Ø D <sub>H</sub> [H6, H7] max. min.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING max. min.	CLEARANCE C <sub>D</sub> max. min.										
6570DP4B	65	70		70.25 69.75		64.970 64.940		70.030 70.000	65.110 64.990																									
7050DP4B	70	7.5	2.505	50.25 49.75	f7	69.970		75.030	70.110	0.170																								
7070DP4B	70	75	2.460	70.25 69.75	1/	69.940		75.000	69.990	0.020																								
7580DP4B	75	80		80.25 79.75		74.970 74.940		80.030 80.000	75.110 74.990																									
8060DP4B	00	0.5		60.50 59.50		80.000		85.035	80.155	0.201																								
80100DP4B	80	85	85	85	85		100.50 99.50		79.946		85.000	80.020	0.020																					
85100DP4B	85	90		100.50 99.50		85.000 84.946		90.035 90.000	85.155 85.020																									
9060DP4B	00	95		0.5	0.5	0.5	0.5	0.5	0.5	0.5	0.5		60.50 59.50		90.000	H7	95.035	90.155																
90100DP4B	90		2.490	100.50 99.50	l. 0	89.946		95.000	90.020																									
95100DP4B	95	100	2.440	100.50 99.50	h8	95.000 94.946		100.035 100.000	95.155 95.020	0.209																								
10060DP4B	100	105					105		405		60.50 59.50		100.000		105.035	100.155	0.020																	
100115DP4B	100	105		115.50 114.50		99.946		105.000	100.020																									
105115DP4B	105	110		115.50 114.50		105.000 104.946		110.035 110.000	105.155 105.020																									
110115DP4B	110	115		115.50 114.50		110.000 109.946		115.035 115.000	115.155 115.020																									

## 8.10 DP4-B FLANGED BUSHES - metric sizes



Dimensions and Tolerances according to ISO 3547 and GGB Specifications

#### **OUTSIDE Co AND INSIDE Ci CHAMFERS**

WALL THICKNESS S <sub>3</sub>	C₀ MACHINED		C <sub>i</sub> (b)
0.75	$0.5 \pm 0.3$	$0.5 \pm 0.3$	-0.1 to -0.4
1.5	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.7
1	$0.6 \pm 0.4$	$0.6 \pm 0.4$	-0.1 to -0.5

WALL THICKNESS S <sub>3</sub>	C <sub>o</sub> (a) MACHINED / ROLLEI		C <sub>i</sub> (b)
2	$1.2 \pm 0.4$	$1.0 \pm 0.4$	-0.1 to -0.7
2.5	$1.8 \pm 0.6$	$1.2 \pm 0.4$	-0.2 to -1.0

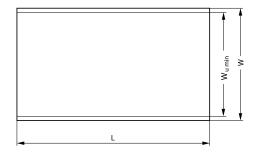
- (a) = chamfer  $C_o$  machined or rolled at the opinion of the manufacturer
- (b) =  $C_i$  can be a radius or a chamfer in accordance with ISO 13715

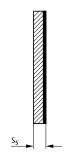
PART NO.		IINAL METER D <sub>o</sub>	WALL THICK- NESS S <sub>3</sub> max. min.	FLANGE THICKN. S <sub>fl</sub> max. min.	FLANGE Ø D <sub>fl</sub> max. min.	WIDTH B max. min.	D <sub>J</sub> [	SHAFT Ø H6, F7, H8] max. min.	HC D <sub>H</sub>	USING Ø [H6, H7] max. min.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING max. min.	CLEARANCE C <sub>D</sub> max. min.					
BB0304DP4B	3	4.5	0.750	0.80	7.50 6.50	4.25 3.75	h6	3.000 2.994	H6	4.508 4.500	3.048 3.000	0.054 0.000					
BB0404DP4B	4	5.5	0.730	0.70	9.50 8.50	4.25 3.75	no	4.000 3.992	ПО	5.508 4.500	4.048 4.000	0.056 0.000					
BB0505DP4B	5	7			10.50 9.50	5.25 4.75		4.990 4.978		7.015 7.000	5.055 4.990	0.077 0.000					
BB0604DP4B					12.50	4.25 3.75		5.990		8.015	6.055	0.077					
BB0608DP4B	6	8		11.50	8.25 7.75		5.978		8.000	5.990	0.000						
BB0806DP4B		4.0			15.50	5.75 5.25		7.987		10.015	8.055	0.083					
BB0810DP4B	8	10			14.50	14.50	9.75 9.25		7.972	2	10.000	7.990	0.003				
BB1007DP4B	10	4.0							18	18.50	7.25 6.75		9.987		12.018	10.058	0.086
BB1012DP4B	10	12				17.50	12.25 11.75		9.972		12.000	9.990	0.003				
BB1207DP4B		14	14	14	14	14		1.005 0.980	1.05 0.80		7.25 6.75	f7		H7			
BB1209DP4B	12								20.50 19.50	9.25 8.75		11.984 11.966		14.018 14.000	12.058 11.990		
BB1212DP4B									13.30	12.25 11.75		11.500		1 1.000	11.550		
BB1417DP4B	14	16			22.50 21.50	17.25 16.75		13.984 13.966		16.018 16.000	14.058 13.990	0.092					
BB1512DP4B	4.5	4.7			23.50	12.25 11.75		14.984		17.018	15.058	0.006					
BB1517DP4B	15	17			22.50 24.50 23.50	22.50	17.25 16.75		14.966		17.000	14.990					
BB1612DP4B		4.0				24.50	12.25 11.75		15.984		18.018	16.058					
BB1617DP4B	16	18				17.25 16.75		15.966		18.000	15.990						

PART NO.		IINAL METER D <sub>o</sub>	WALL THICK- NESS S <sub>3</sub> max. min.	FLANGE THICKN. S <sub>fl</sub> max. min.	FLANGE Ø D <sub>fl</sub> max. min.	WIDTH B max. min.		SHAFT Ø H6, F7, H8] max. min.		USING Ø [H6, H7] max. min.	BUSH Ø D <sub>i,a</sub> ASSEMBLY IN H6/H7 HOUSING max. min.	CLEARANCE C <sub>D</sub> max. min.								
BB1812DP4B	10	20	1.005	1.05	26.50	12.25 11.75		17.984		20.021	18.061	0.095								
BB1822DP4B	18	20	0.980	0.80	25.50	22.25 21.75		17.966		20.000	17.990	0.006								
BB2012DP4B	20	22			30.50 29.50	11.75 11.25		19.980		23.021	20.071									
BB2017DP4B	20	22	1.505	1.60		16.75 16.25		19.959		23.000	19.990	0.112								
BB2512DP4B	25	20	20	20	20	20	20	20	20	20	1.475	1.30	35.50	11.75 11.25		24.980		28.021	25.071	0.010
BB2522DP4B	25	28			34.50	21.75 21.25	f7	24.959 H7	H7	H7 28.000	24.990									
BB3016DP4B	20	34			42.50	16.25 15.75		29.980		34.025	30.085	0.126								
BB3026DP4B	30	34	2.005	2.10	26.25 25.75		29.959		34.000	29.990	0.010									
BB3526DP4B	35	39	1.970	1.80	47.50 46.50	26.25 25.75		34.975 34.950		39.025 39.000	35.085 34.990	0.135 0.015								
BB4026DP4B	40	44			53.50 52.50	26.25 25.75		39.975 39.950		44.025 44.000	40.085 39.990	0.135 0.015								
BB4526DP4B	45	50	2.505 2.460	2.60 2.30	58.50 57.50	26.25 25.75		44.975 44.950		50.025 50.000	45.105 44.990	0.155 0.015								

## 8.11 DP4-B PLATES - metric sizes







PART NO.	LENGTH L max. min.	TOTAL WIDTH W	USABLE WIDTH WU min	THICKNESS S <sub>S</sub> max. min.
S07085DP4B		95	85	0.74 0.70
S10180DP4B	503 500		100	0.74 0.70
S15180DP4B				1.52 1.48
S20180DP4B			180	1.98 1.94
S25180DP4B				2.46 2.42

## 9 Test Methods

#### 9.1 MEASUREMENT OF WRAPPED BUSHES

It is not possible to accurately measure the external and internal diameters of a wrapped bush in the free condition. In its free state a wrapped bush will not be perfectly cylindrical and the butt joint may be open. When correctly installed in a housing the butt joint will be tightly closed and the bush will conform to the housing. For this reason the external diameter and internal diameter of a wrapped bush can only be checked with special gauges and test equipment.

The checking methods are defined in ISO 3547 Parts 1 to 7.

#### **TEST A OF ISO 3547 PART 2**

Checking the external diameter in a test machine with checking blocks and adjusting mandrel.

TEST A OF ISO 3547 PART 2 ON 2015DP4°	
Checking block and setting mandrel d <sub>ch,1</sub>	23.062 mm
Test force F <sub>ch</sub>	4500 N
Limits for $\Delta z$	0 and -0.065 mm
Bush Outside diameter Do	23.035 to 23.075 mm

Table 18: Test A of ISO 3547 Part 2

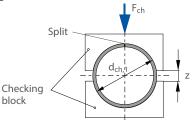


Fig. 40: Test A, data for drawing

#### **TEST B (ALTERNATIVELY TO TEST A)**

Check external diameter with GO and NO GO ring gauges.

#### TEST C

Checking the internal diameter of a bush pressed into a ring gauge, which nominal diameter corresponds to the dimension specified in table 6 of ISO 3547 Part 2 (Example  $D_i = 20$  mm).

#### MEASUREMENT OF WALL THICKNESS (ALTERNATIVELY TO TEST C)

The wall thickness is measured at one, two or three positions axially according to the bearing dimensions.

B [mm]	X [MM]	MEASUREMENT POSITION
≤15	B/2	1
>15 ≤50	4	2
>50 ≤90	6 and B/2	3
>90	8 and B/2	3

Table 19: Measurement position

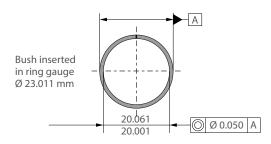


Fig. 41: Test C, data for drawing

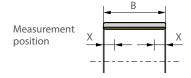


Fig. 42: Test C, measurement position

#### **TEST D**

Check external diameter by precision measuring tape.

# 10 Bearing Application Data Sheet OG



Please complete the form below and share it with your sales engineer.

DATA FOR BEARING DESIGN CALCULATION	N	
Application:		
Project/No.:	Quantity: New [	Design Existing Design
Steady Load Rotating Load	Rotational Movement Oscilla	ating Movement Linear Movement
DIMENSIONS [MM]	FITS & TOLERANCES	BEARING TYPE
Inside diameter D <sub>i</sub>	Shaft D <sub>J</sub>	☐ Cylindrical
Outside diameter $D_o$	Bearing housing D <sub>H</sub>	bush
Length B		
Flange diameter D <sub>fl</sub>	OPERATING ENVIRONMENT	
Flange thickness B <sub>fl</sub>	Ambient temperatureT <sub>amb</sub> [°]	
Wall thickness S <sub>T</sub>	Bearing housing material	
Length of slideplate L	Housing with good heating transfer properties	
Width of slideplate W	Light pressing or insulated housing with poor heat transfer properties	
Thickness of slideplate S <sub>S</sub>	_	Flanged bush
LOAD	Non metal housing with poor heat transfer properties	
☐ Static load	Alternate operation in water and dry	<b>†</b>
☐ Dynamic load	OPERATING ENVIRONMENT	۵ ا ا ا ا ا ا ا ا ا ا ا ا ا ا ا ا ا ا ا
Axial load F [N]		
Radial load F [N]	Dry Continuous lubrication	↓   <u>↓</u>
MOVEMENT	Process fluid lubrication	· vinninini
	Initial lubrication only	
Rotational speed N [1/min]	Hydrodynamic conditions	☐ Thrust washer → KST
Speed U [m/s]	Process fluid	<b>→</b>
Length of stroke L <sub>S</sub> [mm]	Lubricant	<b>A</b>
Frequence of stroke [1/min]	Dynamic viscosity η[mPas]	; <u>-</u>
Oscillating $\phi$ $\phi$ $\phi$ [°] cycle	SERVICE HOURS PER DAY	
<del>(                                    </del>	Continuous operation	
	Intermittent operation	
Osc. frequence N <sub>osz</sub> [1/min]	Operating time	Slide plate
MATING SURFACE	Days per year	
Material		<u> </u>
Hardness HB/HRC	SERVICE LIFE	<sup>†</sup>   <del>&lt; _ </del> _
Surface finish R <sub>a</sub> [µm]	Required service life L <sub>H</sub> [h]	<u> </u>
CUSTOMER INFORMATION		>
		_
Street		— Connected was the
		Special parts (sketch)
•	Fax	
·	Ταλ	
Name		_
Email Address	Date	

# **Formula Symbols and Designations**

SYMBOL	UNIT	DESIGNATION
А	mm²	Surface area of DU® bearing
$A_{M}$	mm <sup>2</sup>	Surface area of mating surface in contact with DU® bearing (slideway)
a <sub>B</sub>	-	Bearing size factor
a <sub>C</sub>	-	Application factor for bore burnishing or machining
a <sub>E</sub>	-	High load factor
a <sub>E1</sub>	-	Specific load factor (slideways)
a <sub>E2</sub>	-	Speed, temperature and material factor (slideways)
a <sub>E3</sub>	-	Relative contact area factor (slideways)
$a_L$	-	Life correction constant
a <sub>M</sub>	-	Mating surface material factor
$a_{T}$	-	Temperature application factor
В	mm	Nominal bush length
C	1/min	Dynamic load frequency
$C_D$	mm	Installed diametrical clearance
$C_{i}$	mm	ID chamfer length
C <sub>o</sub>	mm	OD chamfer length
$C_T$	-	Total number of dynamic load cycles
$D_{C}$	mm	Diameter of burnishing tool
$D_fl$	mm	Nominal bush flange OD
$D_H$	mm	Housing Diameter
D <sub>i</sub>	mm	Nominal bush and thrust washer ID
D <sub>i,a</sub>	mm	Bush ID when assembled in housing
D <sub>J</sub>	mm	Shaft diameter
$D_Nth$	nvt	Max. thermal neutron dose
D <sub>o</sub>	mm	Nominal bush and thrust washer OD
$D_Y$	Gy	Max. Gamma radiation dose
$d_D$	mm	Dowel hole diameter
$d_L$	mm	Oil hole diameter
$d_p$	mm	Pitch circle diameter for dowel hole
F	N	Bearing load
$F_{ch}$	N	Test force
Fi	N	Insertion force
f	-	Coefficient of friction

SYMBOL	UNIT	DESIGNATION
H <sub>a</sub>	mm	Depth of housing recess (e.g. for thrust washers)
H <sub>d</sub>	mm	Diameter of housing recess (e.g. for thrust washers)
L	mm	Strip length
L <sub>H</sub>	h	Bearing service life
$L_S$	mm	Length of stroke (slideway)
N	1/min	Oscillating movement frequency
$N_{\rm osz}$	1/min	Specific load
Р	MPa	Specific load limit
$P_{lim}$	MPa	Maximum static load
P <sub>sta,max</sub>	MPa	Maximum dynamic load
P <sub>dyn,max</sub>	MPa	Permissible number of cycles
Q	-	Rotational speed
$R_a$	μm	Surface roughness (DIN 4768, ISO/DIN 4287/1)
R <sub>OB</sub>	Ω	Electrical resistance
$S_3$	mm	Bush wall thickness
S <sub>fl</sub>	mm	Flange thickness
$S_S$	mm	Strip thickness
$S_T$	mm	Thrust washer thickness
Т	°C	Temperature
$T_{amb}$	°C	Ambient temperature
$T_{\text{max}}$	°C	Maximum temperature
$T_{min}$	°C	Minimum temperature
U	m/s	Sliding speed
W	mm	Strip width
$W_{Umin}$	mm	Minimum usable strip width
$Z_{T}$	-	Total number of cycles
$\alpha_1$	1/10 <sup>6</sup> K	Coefficient of linear thermal expansion parallel to surface
$\alpha_2$	1/10 <sup>6</sup> K	Coefficient of linear thermal expansion normal to surface
$\sigma_{\rm c}$	MPa	Compressive yield strength
λ	W/mK	Thermal conductivity
φ	0	Angular displacement
η	cР	Dynamic viscosity

## **Product Information**

This document is provided to give you the analysis tools or information to assist you in product selection. Product performance is affected by many factors beyond the control of GGB. Therefore, you must validate the suitability and feasibility of all product selections for your applications.

GGB products are sold subject to GGB's Terms of Sale and Delivery, which include our limited warranty and remedy. You can find these here: https://www.ggbearings.com/en/terms-and-conditions, or ask your GGB representative for a copy.

Products are subject to continual development. GGB retains the right to make specification amendments or improvements to the technical data without prior announcement.

#### **DOCUMENT INFORMATION**

Edition 2025. This edition replaces earlier editions which hereby lose their validity.

Every reasonable effort has been made to ensure the accuracy of the information in this writing, but GGB assumes no liability for errors or omissions or for any other reason.

#### **HEALTH AND SAFETY**

GGB is committed to adhering to all U.S., European and international standards and regulations with regard to lead content. We have established internal processes that monitor any changes to existing standards and regulations, and we work collaboratively with customers and distributors to ensure that all requirements are followed. This includes RoHS and REACH guidelines. GGB is committed to operating in an environmentally conscious and safe manner. We follow numerous industry best practices and are committed to meeting or exceeding a variety of internationally recognized standards for emissions control and workplace safety.

Each of our global locations has management systems in place that adhere to IATF 16949, ISO 9001, ISO 14001 and ISO 45001 quality regulations. Our certificates can be found here:

https://www.ggbearings.com/en/company/certificates.

A detailed explanation of our commitment to REACH and RoHS directives can be found at

https://www.ggbearings.com/en/company/reach-rohs.







# Stronger. Together.









#### **GGB NORTH AMERICA**

P.O. Box 189 | 700 Mid Atlantic Parkway USA | Thorofare, New Jersey, 08086 Tel: +1 856 848 3200 www.ggbearings.com

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